

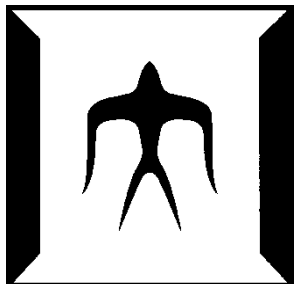
April 20, 2023

Advanced Mechanical Elements (Lecture 2)

*Kinematic analysis of spatial link mechanism
with the systematic kinematic analysis method*
*-Expansion of the systematic kinematic analysis
method to spatial mechanisms -*

Tokyo Institute of Technology
Dept. of Mechanical Engineering
School of Engineering

Prof. Nobuyuki Iwatsuki

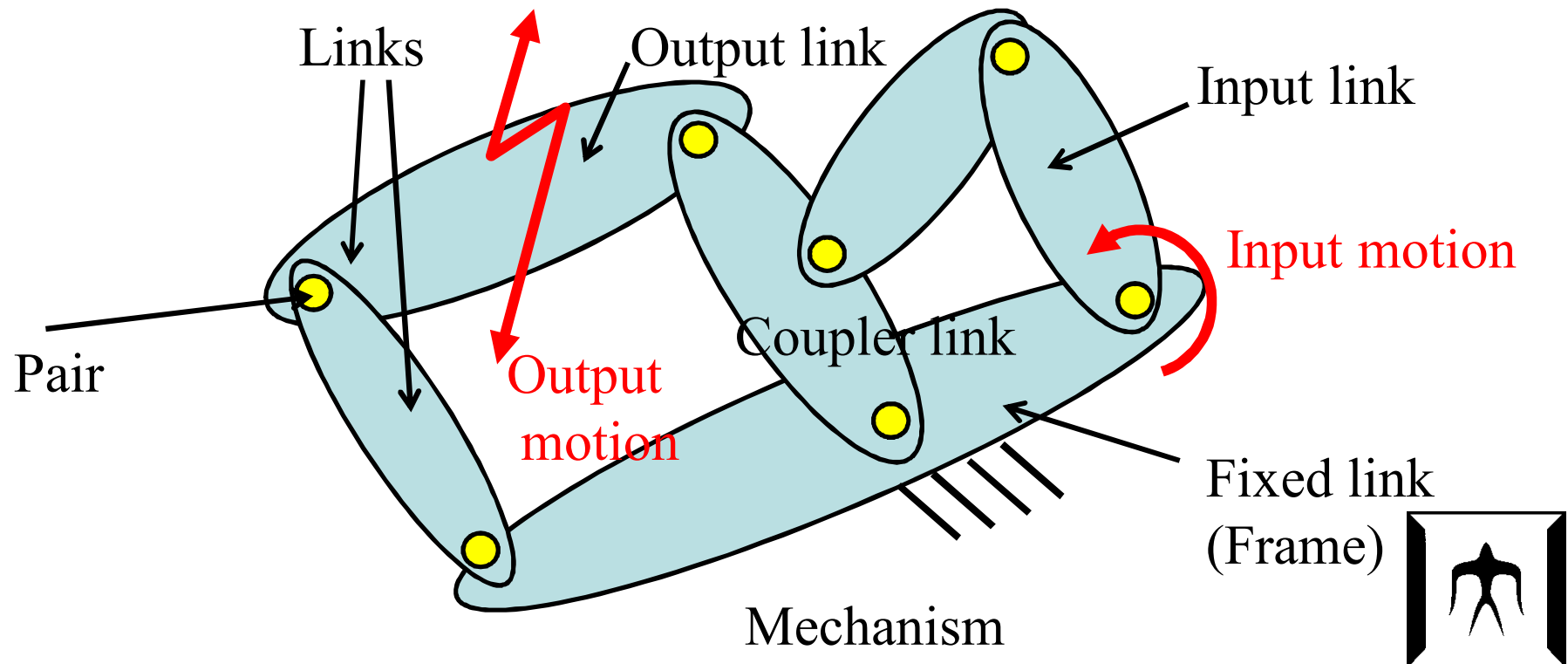


1. Construction and DOF of spatial link mechanisms

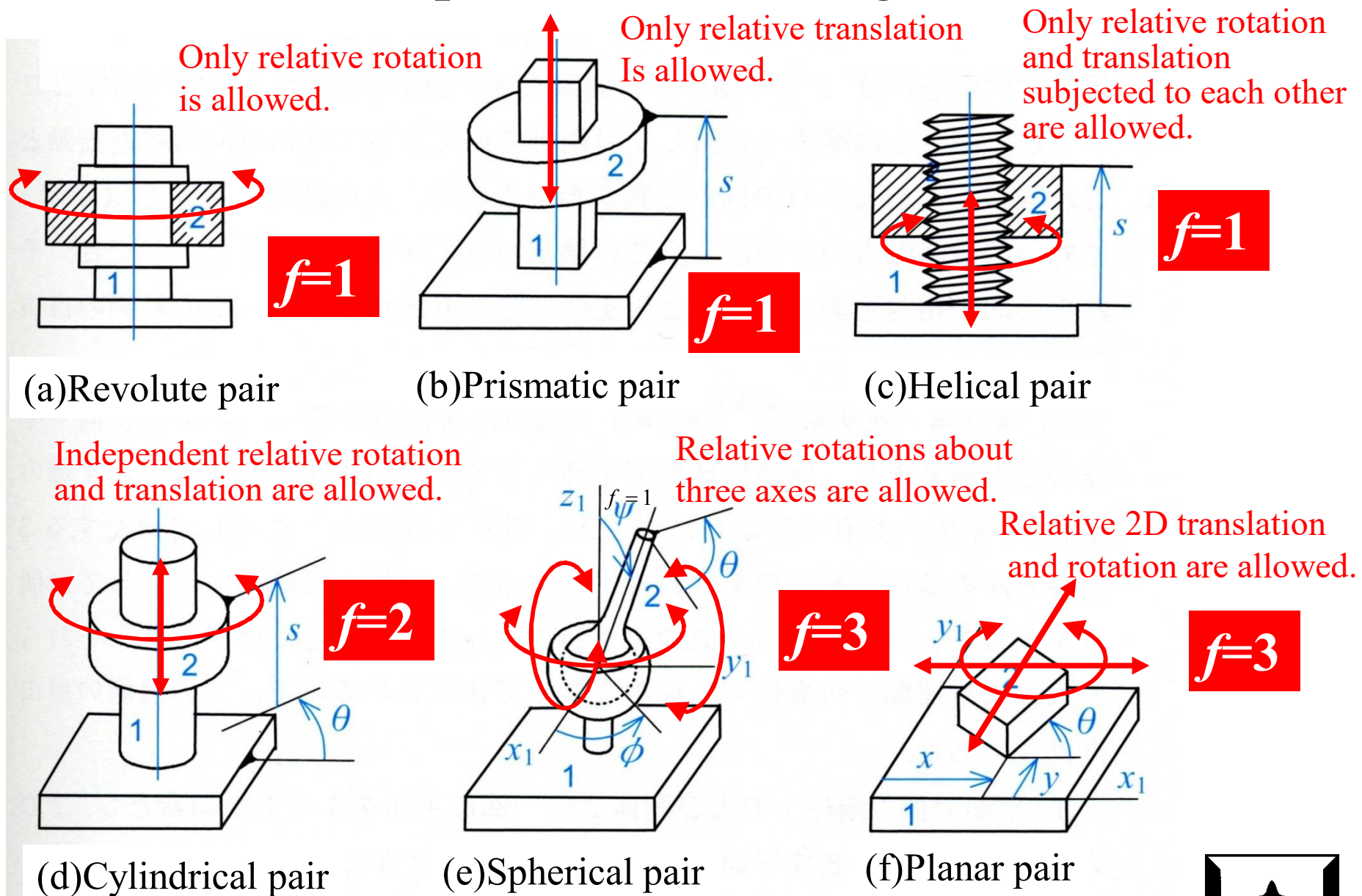
Mechanism and kinematic pairs

Mechanism: One link in a link chain which is composed by many links with **kinematic pairs** should be fixed to frame.

Motions of input links can be transformed to motion of output links.



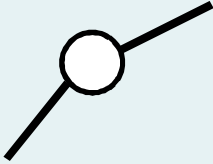
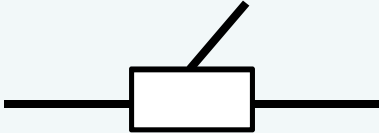
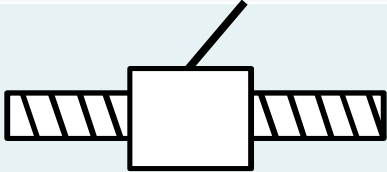
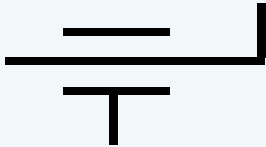
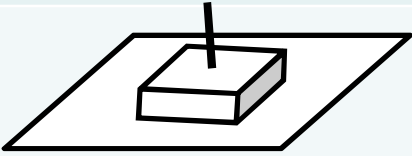
Various kinematic pairs and their degree-of-freedom



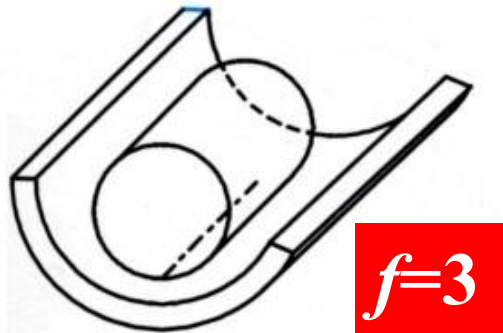
Kinematic pairs with low DOF



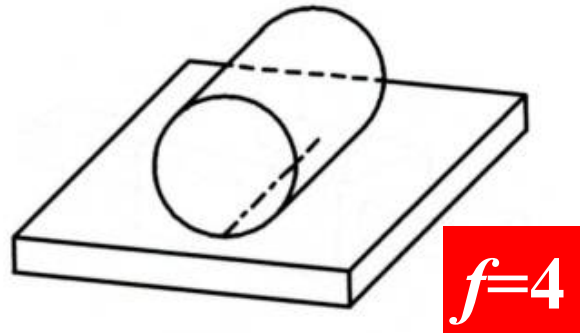
Representation of kinematic pairs with lower DOF

Name of pair	Symbol	Representation in mechanism diagram
Revolute pair	R	
Prismatic pair	P	
Helical pair	H	
Cylindrical pair	C	
Spherical pair	S	
Planar pair	E	

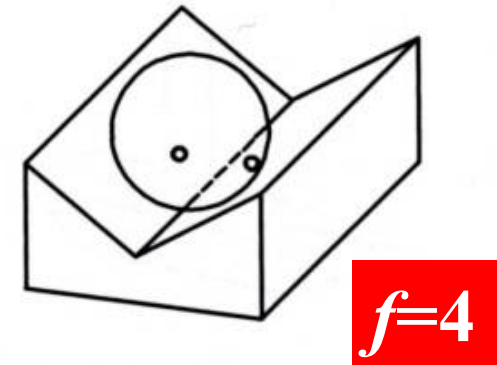




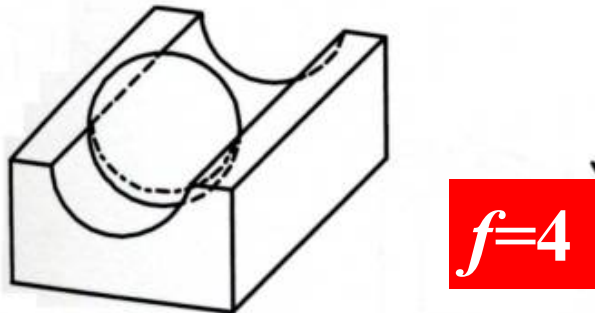
(a)Pillar-cylinder pair



(b)Pillar-plane pair



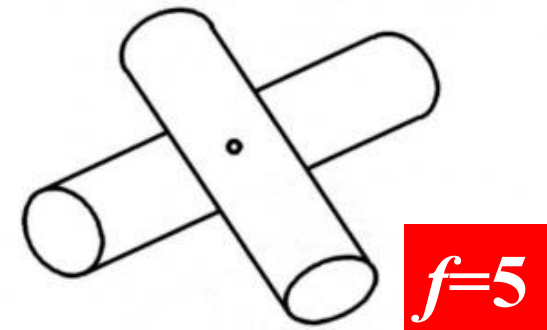
(c)Sphere - V slot pair



(d)Sphere-circular slot pair



(d)Sphere-spherical plane pair



(d)Two pillar pair

They are hardly used in general mechanisms.

Kinematic pairs with high DOF



Degrees-of-Freedom (DOF) of Mechanism

Definition:

Number of kinematic parameters to determine position and posture of all links in the mechanism

Equation of DOF: (Gruebler's Equation)

$$F = F_s(N - 1) - \sum_{f=1}^{F_s-1} (F_s - f)J_f$$

Diagram illustrating the components of Gruebler's Equation for Degrees of Freedom (DOF) of a mechanism:

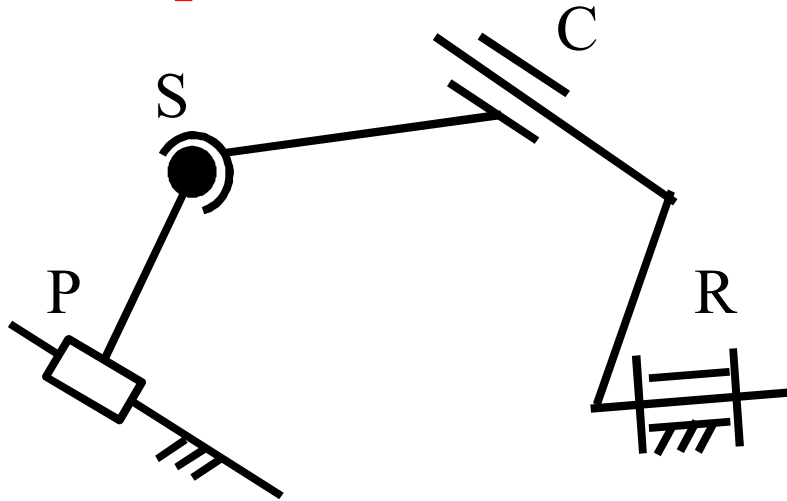
- F : DOF of Mechanisms
- F_s : DOF of space
 - Planar: $F_s=3$
 - Spherical: $F_s=3$
 - Spatial: $F_s=6$
- N : Number of links
- J_f : DOF of pairs
- f : Number of pairs having f DOF

DOF which are constrained by kinematic pairs

Number of parameters to determine position and posture of all moving links which are not constrained with kinematic pairs

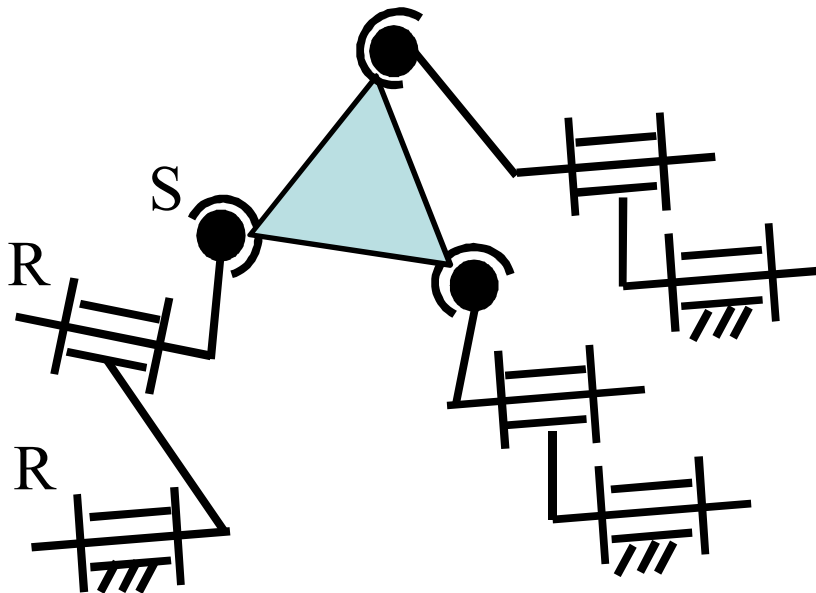


Example of DOF calculation:



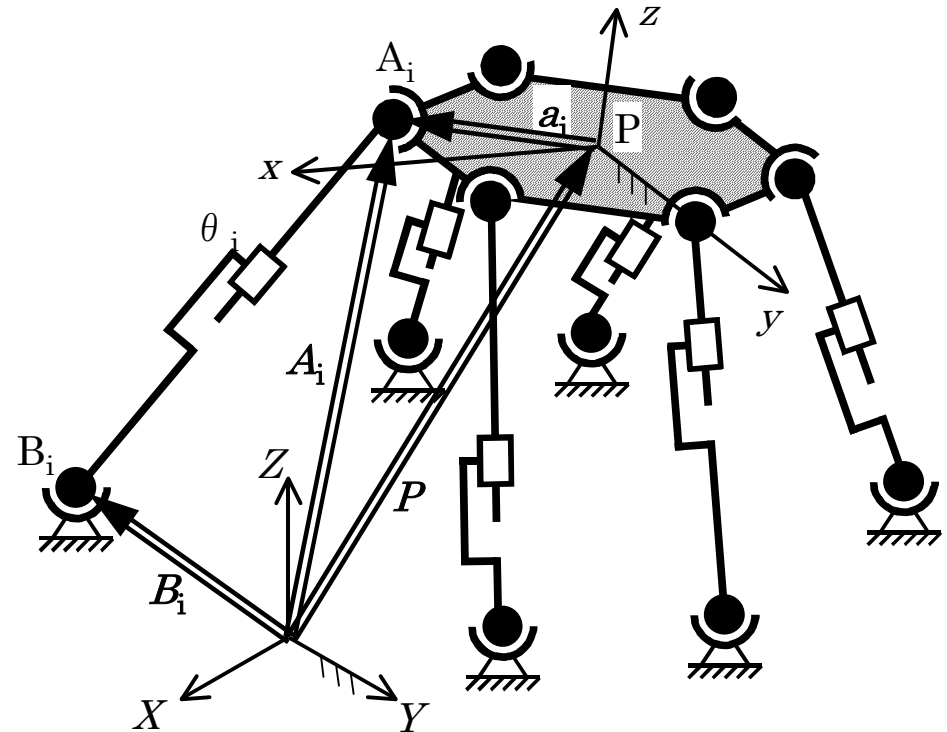
RCSP spatial 4-bar link mechanism

$$F=6(4-1)-(6-1)2-(6-2)-(6-3)=1$$



3RRS spatial manipulator

$$F=6(8-1)-(6-1)6-(6-3)3=3$$



Stewart platform Manipulator

$$F=6(14-1)-(6-1)6-(6-3)12=12$$

**DOF of platform is 6 while having
3 DOF of rotation of S-P-S chain**

2. Understanding of Three-dimensional angular motion

“You should have studied them in Engineering Dynamics!”

Comparison between planar and spatial motions:

Planar motion

Spatial motion

DOF: 3

6

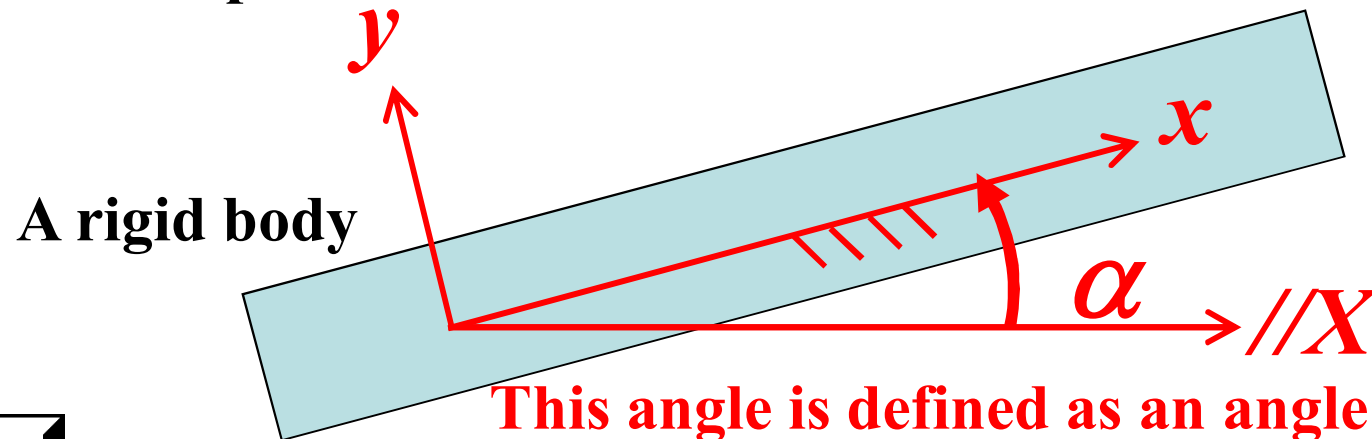
Translational: X, Y coordinates of
COG of a rigid body

X, Y, Z coordinates of
COG of a rigid body

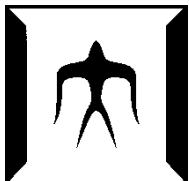
Angular: 1 posture angle

3 posture angles?

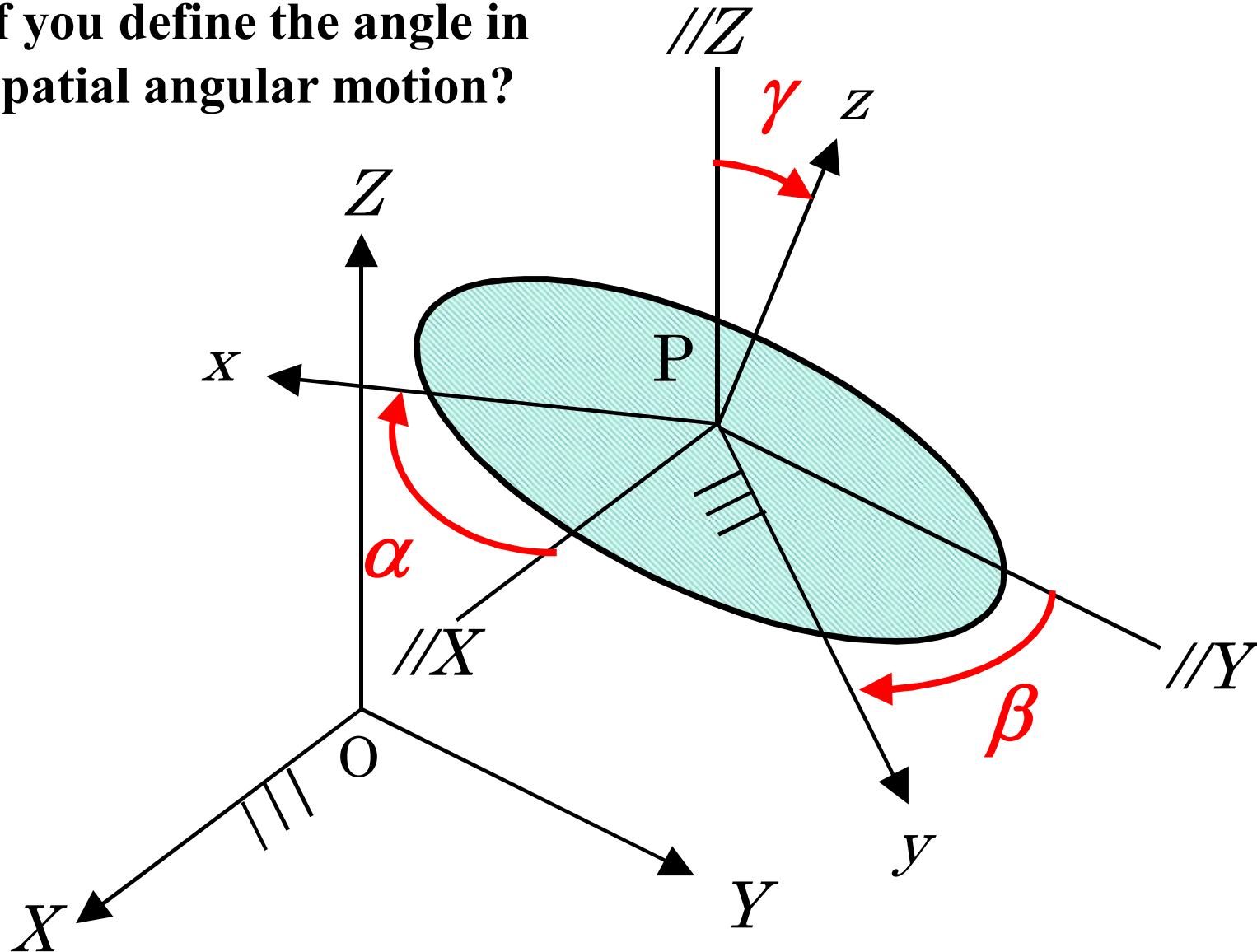
Correct expression



**This angle is defined as an angle between
axes of moving coordinate system fixed on
The rigid body and of fixed coordinate system**



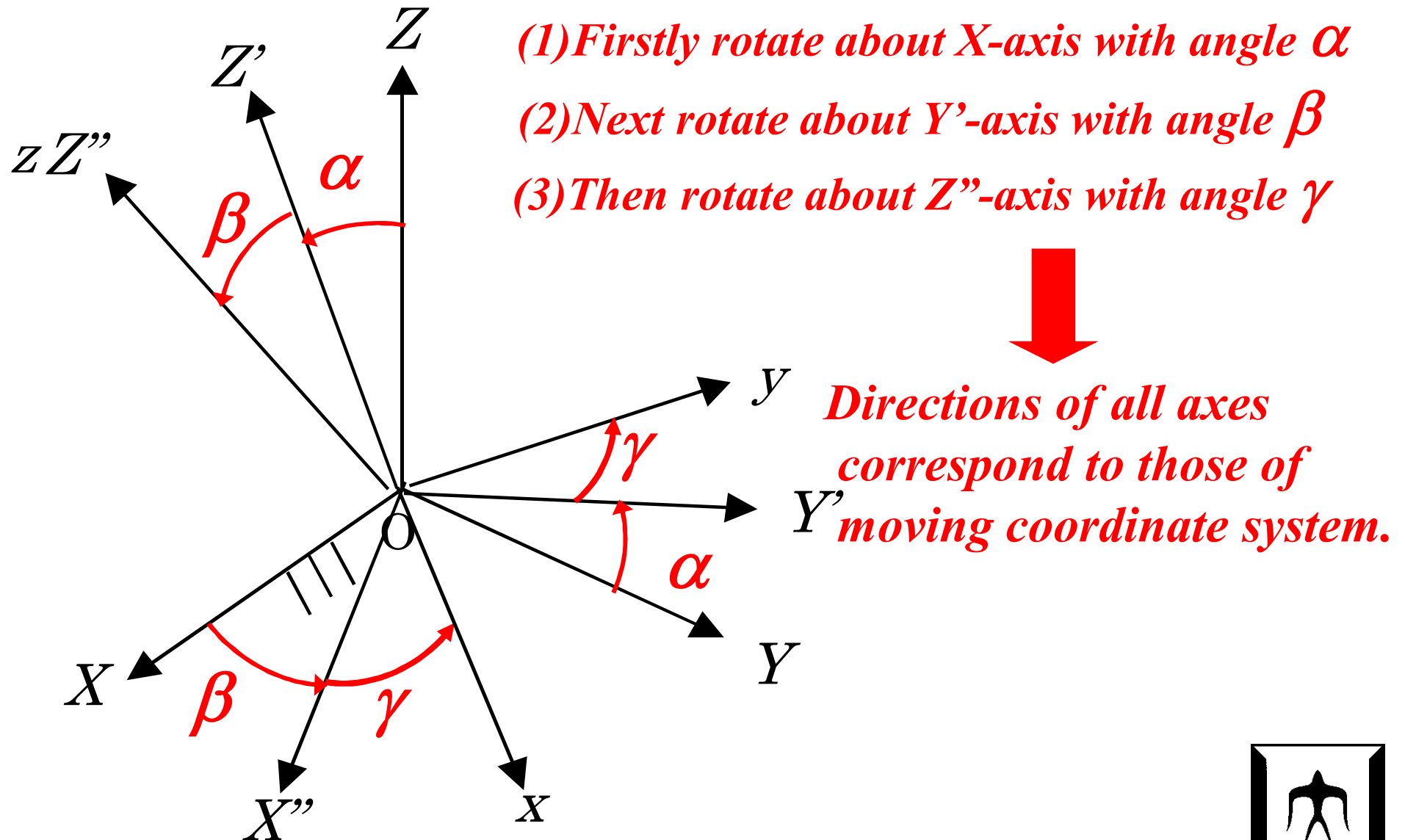
If you define the angle in
Spatial angular motion?



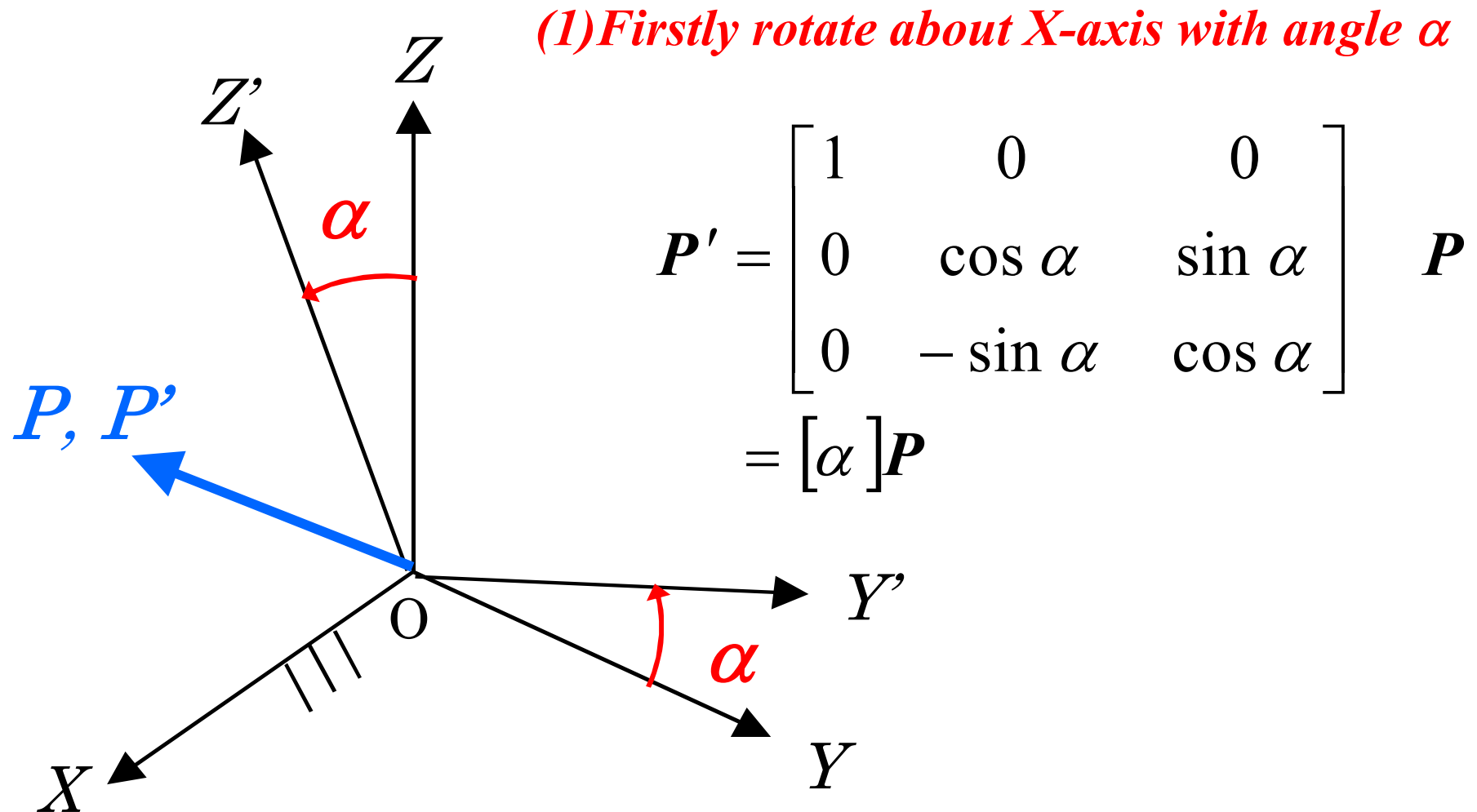
*It is possible to define posture
angles. However it is unavailable!*



Therefore we use the angles to rotate coordinate axis
(Roll-,Pitch-,Yaw-angles)

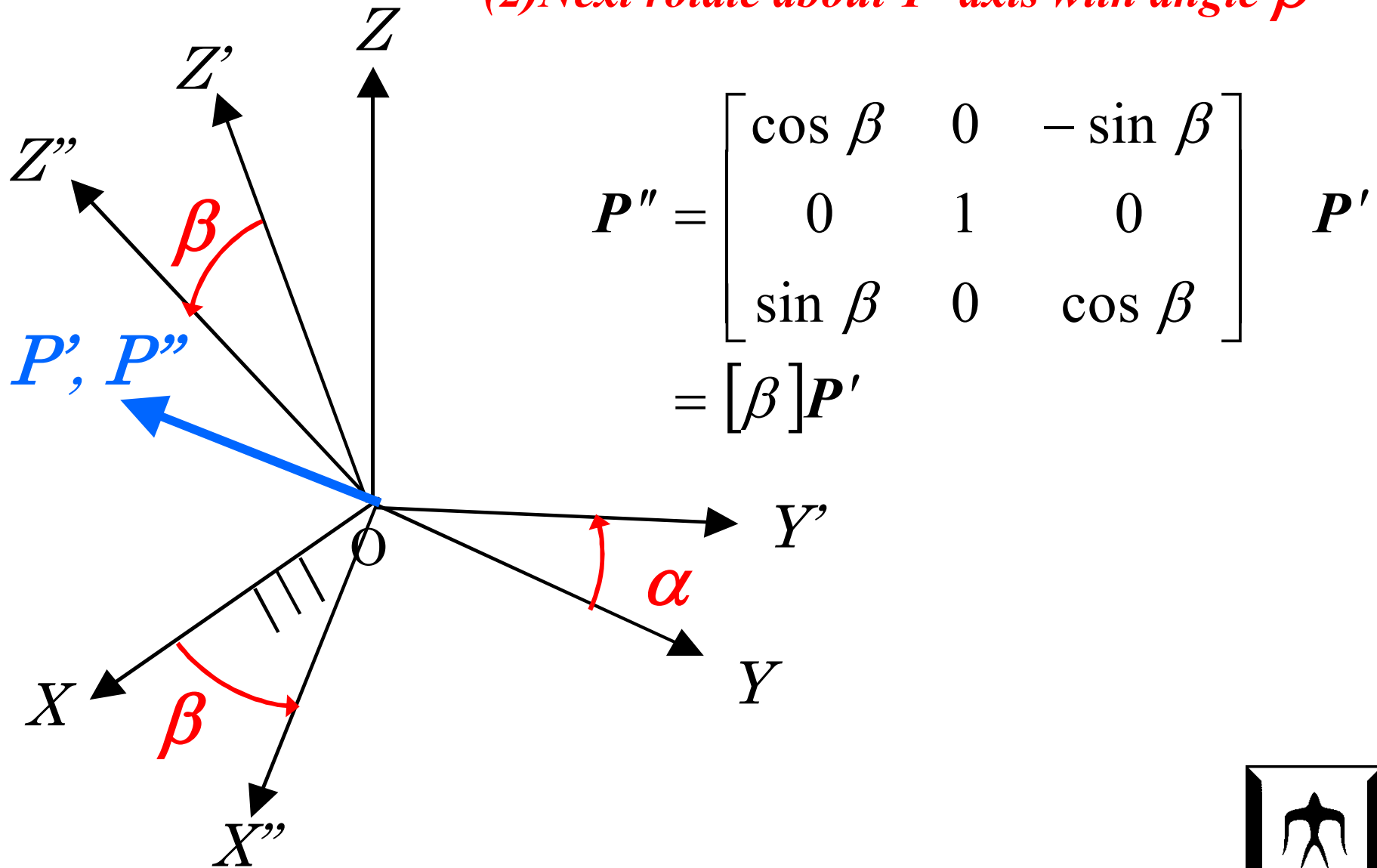


Let focus on a position vector on two coordinate systems



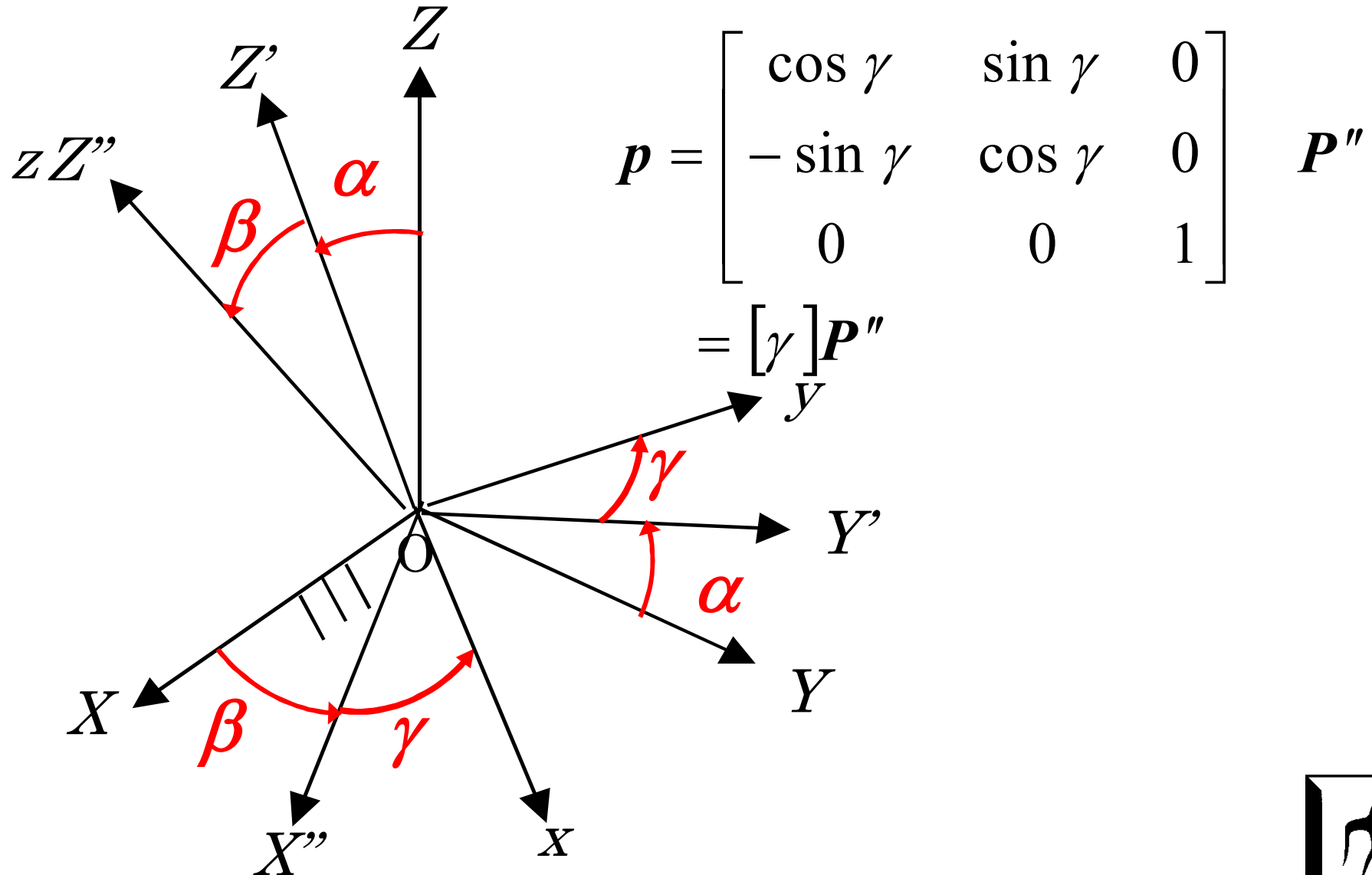
Let focus on a position vector on two coordinate systems

(2) Next rotate about Y' -axis with angle β



Let focus on a position vector on two coordinate systems

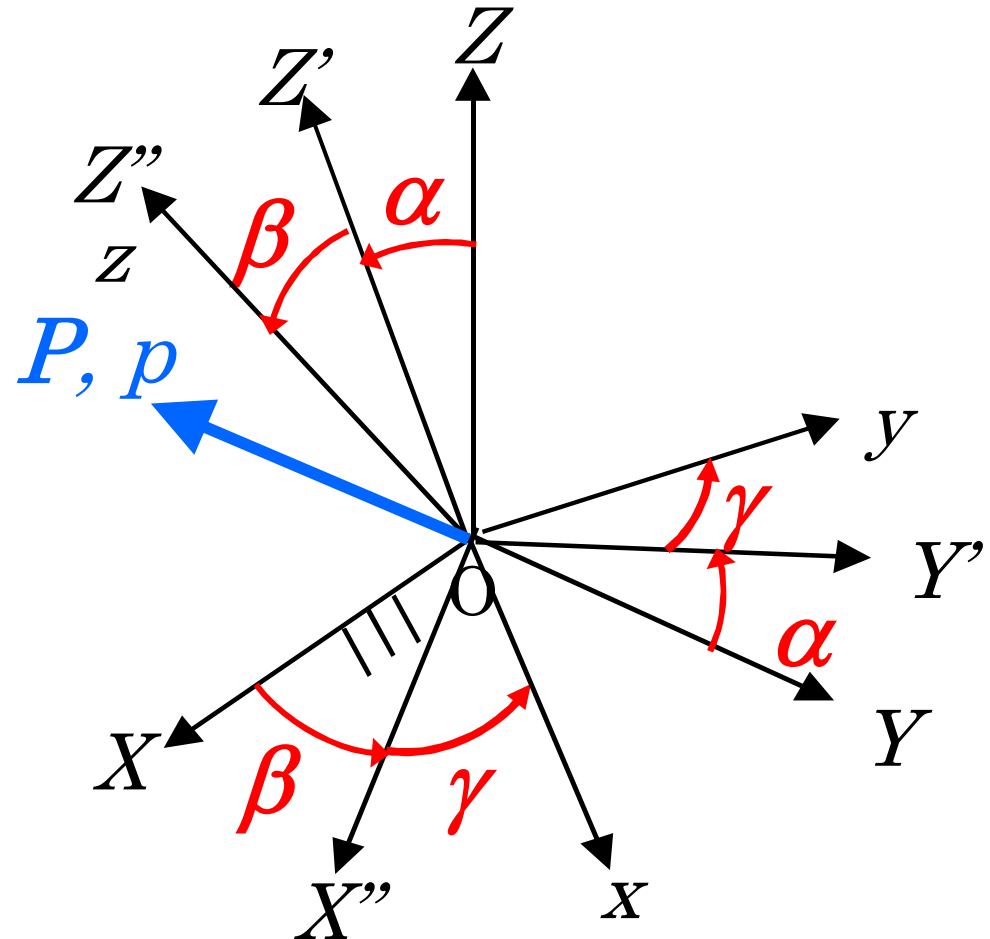
(3) Then rotate about Z'' -axis with angle γ



Resultantly

$$\begin{aligned}
 \mathbf{p} &= \begin{bmatrix} \cos \gamma & \sin \gamma & 0 \\ -\sin \gamma & \cos \gamma & 0 \\ 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} \cos \beta & 0 & -\sin \beta \\ 0 & 1 & 0 \\ \sin \beta & 0 & \cos \beta \end{bmatrix} \begin{bmatrix} 1 & 0 & 0 \\ 0 & \cos \alpha & \sin \alpha \\ 0 & -\sin \alpha & \cos \alpha \end{bmatrix} \mathbf{P} \\
 &= [\gamma][\beta][\alpha]\mathbf{P} \\
 &= [T(\alpha, \beta, \gamma)]\mathbf{P}
 \end{aligned}$$

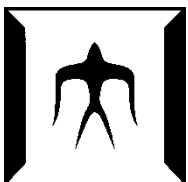
Coordinate transformation matrix



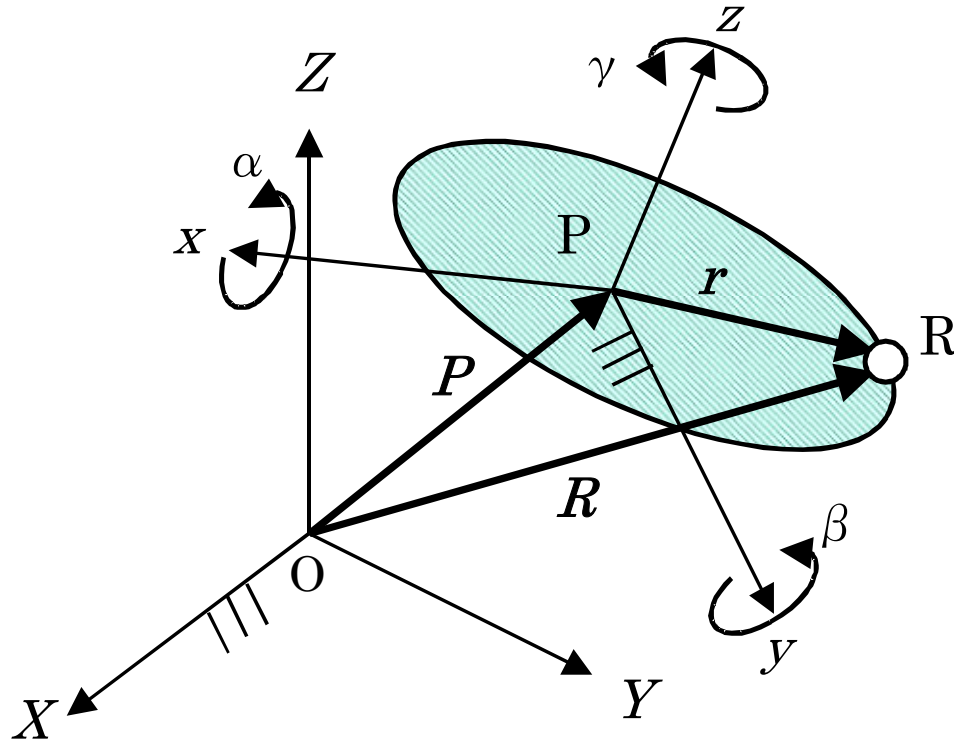
Inverse transformation

$$\begin{aligned} \mathbf{P} &= [T(\alpha, \beta, \gamma)]^{-1} \mathbf{p} \\ &= [\alpha]^{-1} [\beta]^{-1} [\gamma]^{-1} \mathbf{p} \\ &= \begin{bmatrix} 1 & 0 & 0 \\ 0 & \cos \alpha & \sin \alpha \\ 0 & -\sin \alpha & \cos \alpha \end{bmatrix}^{-1} \begin{bmatrix} \cos \beta & 0 & -\sin \beta \\ 0 & 1 & 0 \\ \sin \beta & 0 & \cos \beta \end{bmatrix}^{-1} \begin{bmatrix} \cos \gamma & \sin \gamma & 0 \\ -\sin \gamma & \cos \gamma & 0 \\ 0 & 0 & 1 \end{bmatrix}^{-1} \mathbf{p} \\ &= \begin{bmatrix} 1 & 0 & 0 \\ 0 & \cos \alpha & -\sin \alpha \\ 0 & \sin \alpha & \cos \alpha \end{bmatrix} \begin{bmatrix} \cos \beta & 0 & \sin \beta \\ 0 & 1 & 0 \\ -\sin \beta & 0 & \cos \beta \end{bmatrix} \begin{bmatrix} \cos \gamma & -\sin \gamma & 0 \\ \sin \gamma & \cos \gamma & 0 \\ 0 & 0 & 1 \end{bmatrix} \mathbf{p} \\ &= [\alpha]^T [\beta]^T [\gamma]^T \mathbf{p} \\ &= [T(\alpha, \beta, \gamma)]^T \mathbf{p} \end{aligned}$$

*Inverse matrix can be given
as transposed matrix.*



Coordinate transformation from moving system on the rigid body to fixed system can be expressed as



$$\mathbf{R} = [\mathbf{T}]^T \mathbf{r} + \mathbf{P}$$

$$\dot{\mathbf{R}} = [\dot{\mathbf{T}}]^T \mathbf{r} + \dot{\mathbf{P}}$$

$$\ddot{\mathbf{R}} = [\ddot{\mathbf{T}}]^T \mathbf{r} + \ddot{\mathbf{P}}$$

Spatial angular motion can be expressed with the coordinate transformation matrix.

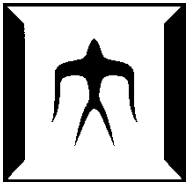


Differentiation of transformation matrix:

$$[\dot{T}]^T = [\dot{\alpha}]^T [\beta]^T [\gamma]^T + [\alpha]^T [\dot{\beta}]^T [\gamma]^T + [\alpha]^T [\beta]^T [\dot{\gamma}]^T$$

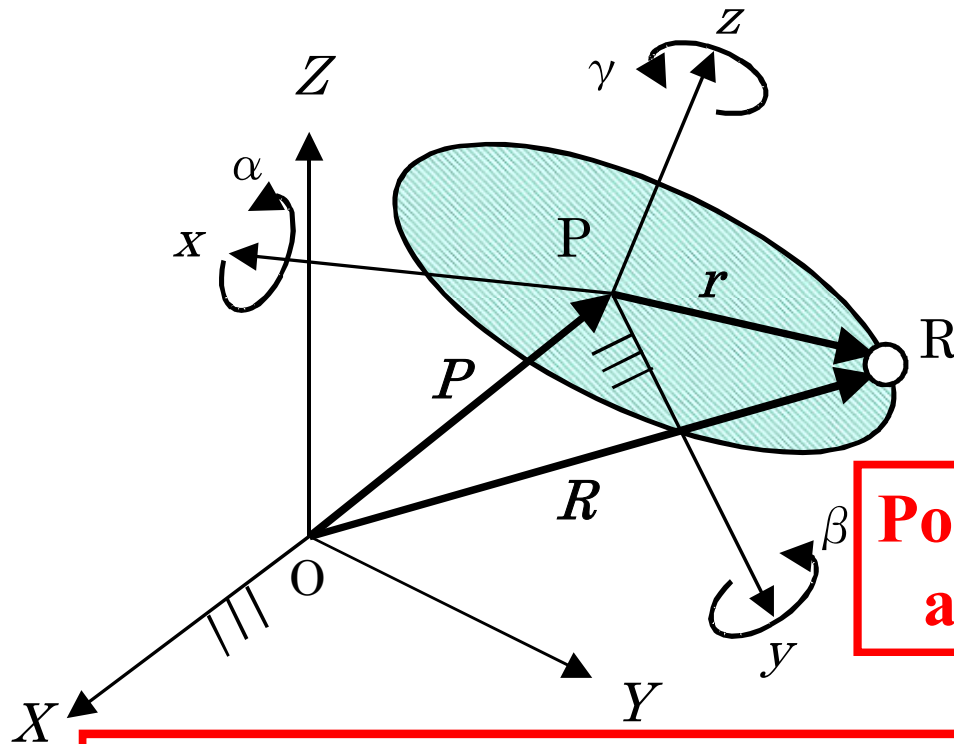
$$\begin{aligned} [\ddot{T}]^T &= [\ddot{\alpha}]^T [\beta]^T [\gamma]^T + [\alpha]^T [\ddot{\beta}]^T [\gamma]^T + [\alpha]^T [\beta]^T [\ddot{\gamma}]^T \\ &\quad + 2[\dot{\alpha}]^T [\dot{\beta}]^T [\gamma]^T + 2[\alpha]^T [\dot{\beta}]^T [\dot{\gamma}]^T + 2[\dot{\alpha}]^T [\beta]^T [\dot{\gamma}]^T \end{aligned}$$

$$\begin{aligned} [\dot{\alpha}]^T &= \begin{bmatrix} 0 & 0 & 0 \\ 0 & -\dot{\alpha} \sin \alpha & -\dot{\alpha} \cos \alpha \\ 0 & \dot{\alpha} \cos \alpha & -\dot{\alpha} \sin \alpha \end{bmatrix} & [\ddot{\alpha}]^T &= \begin{bmatrix} 0 & 0 & 0 \\ 0 & -\ddot{\alpha} \sin \alpha - \dot{\alpha}^2 \cos \alpha & -\ddot{\alpha} \cos \alpha + \dot{\alpha}^2 \sin \alpha \\ 0 & \ddot{\alpha} \cos \alpha - \dot{\alpha}^2 \sin \alpha & -\ddot{\alpha} \sin \alpha - \dot{\alpha}^2 \cos \alpha \end{bmatrix} \\ [\dot{\beta}]^T &= \begin{bmatrix} -\dot{\beta} \sin \beta & 0 & \dot{\beta} \cos \beta \\ 0 & 0 & 0 \\ -\dot{\beta} \cos \beta & 0 & -\dot{\beta} \sin \beta \end{bmatrix} & [\ddot{\beta}]^T &= \begin{bmatrix} -\ddot{\beta} \sin \beta - \dot{\beta}^2 \cos \beta & 0 & \ddot{\beta} \cos \beta - \dot{\beta}^2 \sin \beta \\ 0 & 0 & 0 \\ -\ddot{\beta} \cos \beta + \dot{\beta}^2 \sin \beta & 0 & -\ddot{\beta} \sin \beta - \dot{\beta}^2 \cos \beta \end{bmatrix} \\ [\dot{\gamma}]^T &= \begin{bmatrix} -\dot{\gamma} \sin \gamma & -\dot{\gamma} \cos \gamma & 0 \\ \dot{\gamma} \cos \gamma & -\dot{\gamma} \sin \gamma & 0 \\ 0 & 0 & 0 \end{bmatrix} & [\ddot{\gamma}]^T &= \begin{bmatrix} -\ddot{\gamma} \sin \gamma - \dot{\gamma}^2 \cos \gamma & -\ddot{\gamma} \cos \gamma + \dot{\gamma}^2 \sin \gamma & 0 \\ \ddot{\gamma} \cos \gamma - \dot{\gamma}^2 \sin \gamma & -\ddot{\gamma} \sin \gamma - \dot{\gamma}^2 \cos \gamma & 0 \\ 0 & 0 & 0 \end{bmatrix} \end{aligned}$$



3. Systematic kinematic analysis of spatial link mechanisms

(1) Coordinate transformation matrix and motion of coupler point



$$R = [T]^T r + P$$

$$\dot{R} = [\dot{T}]^T r + \dot{P}$$

$$\ddot{R} = [\ddot{T}]^T r + \ddot{P}$$

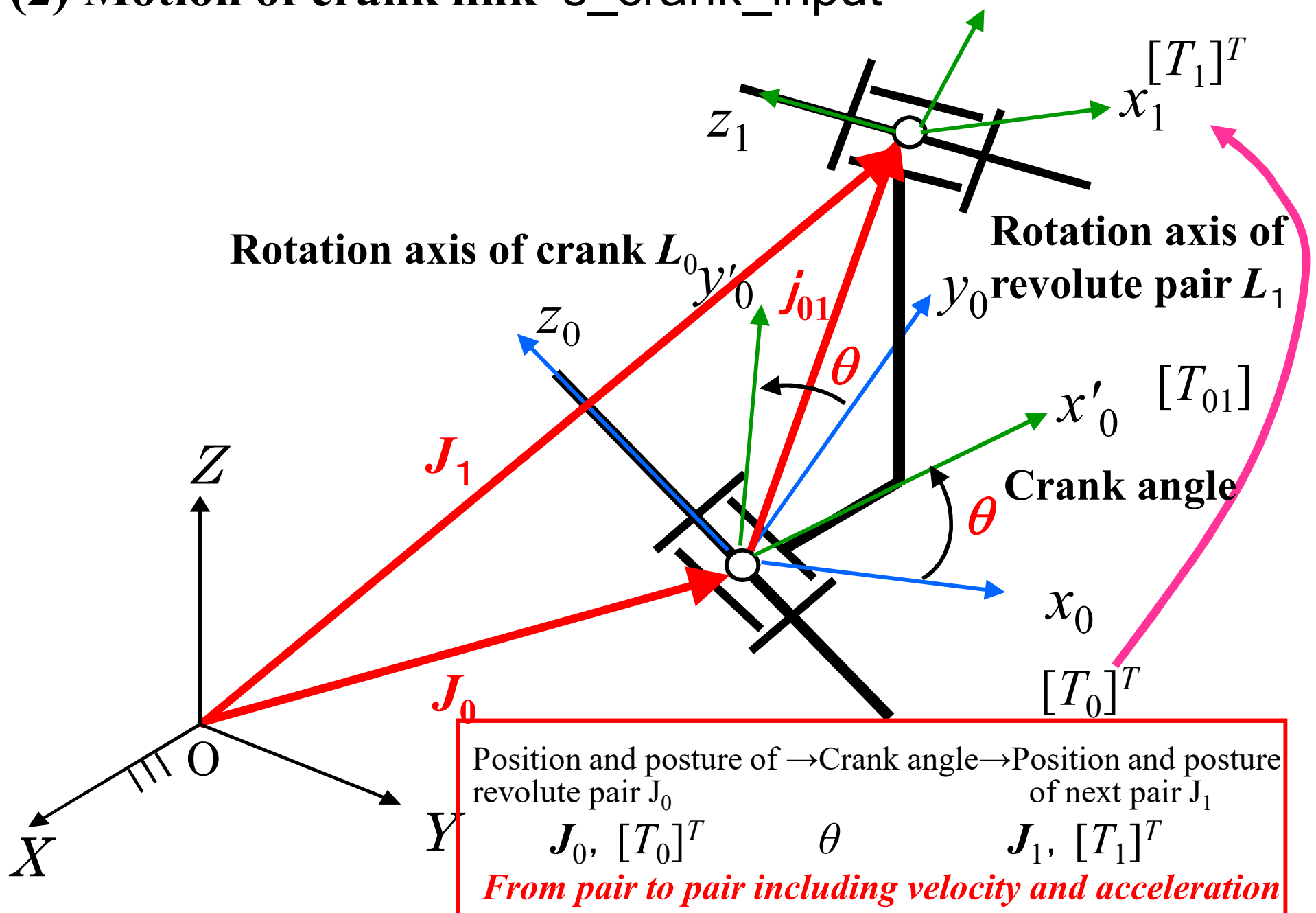
**Point on a spatially moving link:
analysis program: Trans**

Coordinate transformation matrix and posture angles

$$\alpha, \beta, \gamma \iff [T]^T$$

analysis program: RPYTT,TTRPY

(2) Motion of crank link s_crank_input



Equations to calculate:

$$\mathbf{J}_1 = \mathbf{J}_0 + [T_0]^T \begin{bmatrix} \cos \theta & -\sin \theta & 0 \\ \sin \theta & \cos \theta & 0 \\ 0 & 0 & 1 \end{bmatrix} \mathbf{j}_{01} = \mathbf{J}_0 + [T_0]^T [\theta]^T \mathbf{j}_{01}$$

$$\dot{\mathbf{J}}_1 = \dot{\mathbf{J}}_0 + ([\dot{T}_0]^T [\theta]^T + [T_0]^T [\dot{\theta}]^T) \mathbf{j}_{01}$$

$$\ddot{\mathbf{J}}_1 = \ddot{\mathbf{J}}_0 + ([\ddot{T}_0]^T [\theta]^T + 2[\dot{T}_0]^T [\dot{\theta}]^T + [T_0]^T [\ddot{\theta}]^T) \mathbf{j}_{01}$$

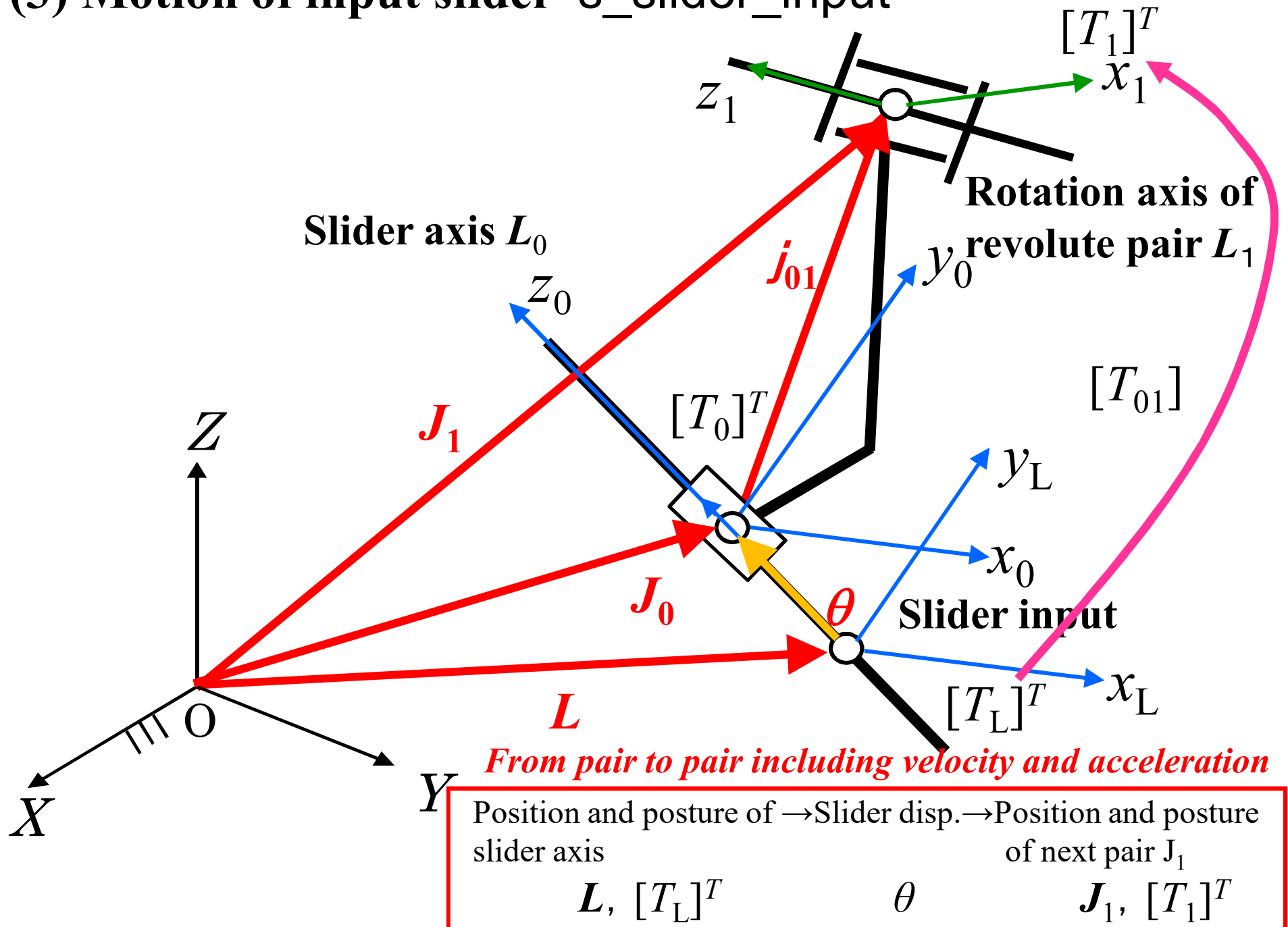
$$[T_1]^T = [T_0]^T [\theta]^T [T_{01}]^T$$

$$[\dot{T}_1]^T = [\dot{T}_0]^T [\theta]^T [T_{01}]^T + [T_0]^T [\dot{\theta}]^T [T_{01}]^T$$

$$[\ddot{T}_1]^T = [\ddot{T}_0]^T [\theta]^T [T_{01}]^T + 2[\dot{T}_0]^T [\dot{\theta}]^T [T_{01}]^T + [T_0]^T [\ddot{\theta}]^T [T_{01}]^T$$



(3) Motion of input slider s_slider_input



Equations to calculate:

$$\mathbf{J}_1 = \mathbf{L} + [\mathbf{T}_L]^T \left(\begin{bmatrix} 0 & 0 & \theta \end{bmatrix}^T + \mathbf{j}_{01} \right)$$

$$\dot{\mathbf{J}}_1 = \dot{\mathbf{L}} + [\dot{\mathbf{T}}_L]^T \left(\begin{bmatrix} 0 & 0 & \theta \end{bmatrix}^T + \mathbf{j}_{01} \right) + [\mathbf{T}_L]^T \begin{bmatrix} 0 & 0 & \dot{\theta} \end{bmatrix}^T$$

$$\begin{aligned} \ddot{\mathbf{J}}_1 = \ddot{\mathbf{L}} + [\ddot{\mathbf{T}}_L]^T \left(\begin{bmatrix} 0 & 0 & \theta \end{bmatrix}^T + \mathbf{j}_{01} \right) + 2[\dot{\mathbf{T}}_L]^T \begin{bmatrix} 0 & 0 & \dot{\theta} \end{bmatrix}^T \\ + [\mathbf{T}_L]^T \begin{bmatrix} 0 & 0 & \ddot{\theta} \end{bmatrix}^T \end{aligned}$$

$$[\mathbf{T}_1]^T = [\mathbf{T}_L]^T [\mathbf{T}_{01}]^T$$

$$[\dot{\mathbf{T}}_1]^T = [\dot{\mathbf{T}}_L]^T [\mathbf{T}_{01}]^T$$

$$[\ddot{\mathbf{T}}_1]^T = [\ddot{\mathbf{T}}_L]^T [\mathbf{T}_{01}]^T$$

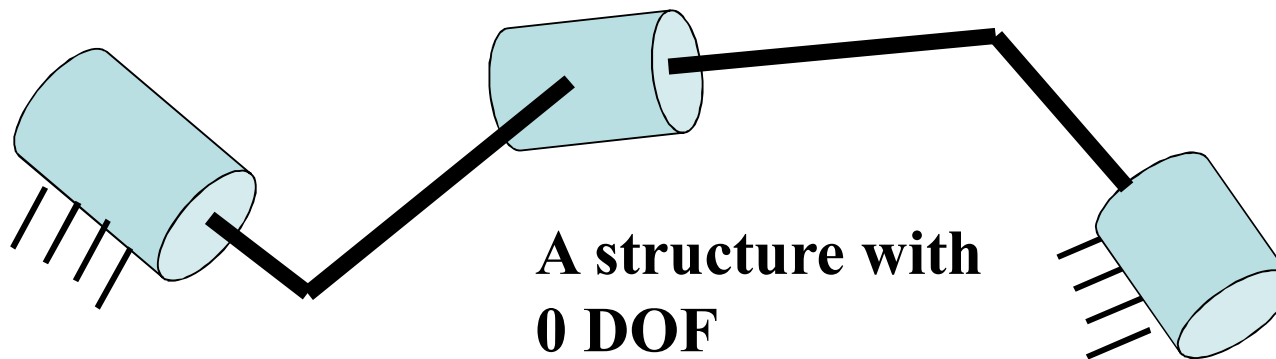
$$\therefore [\mathbf{T}_1] = [\mathbf{T}_L]$$

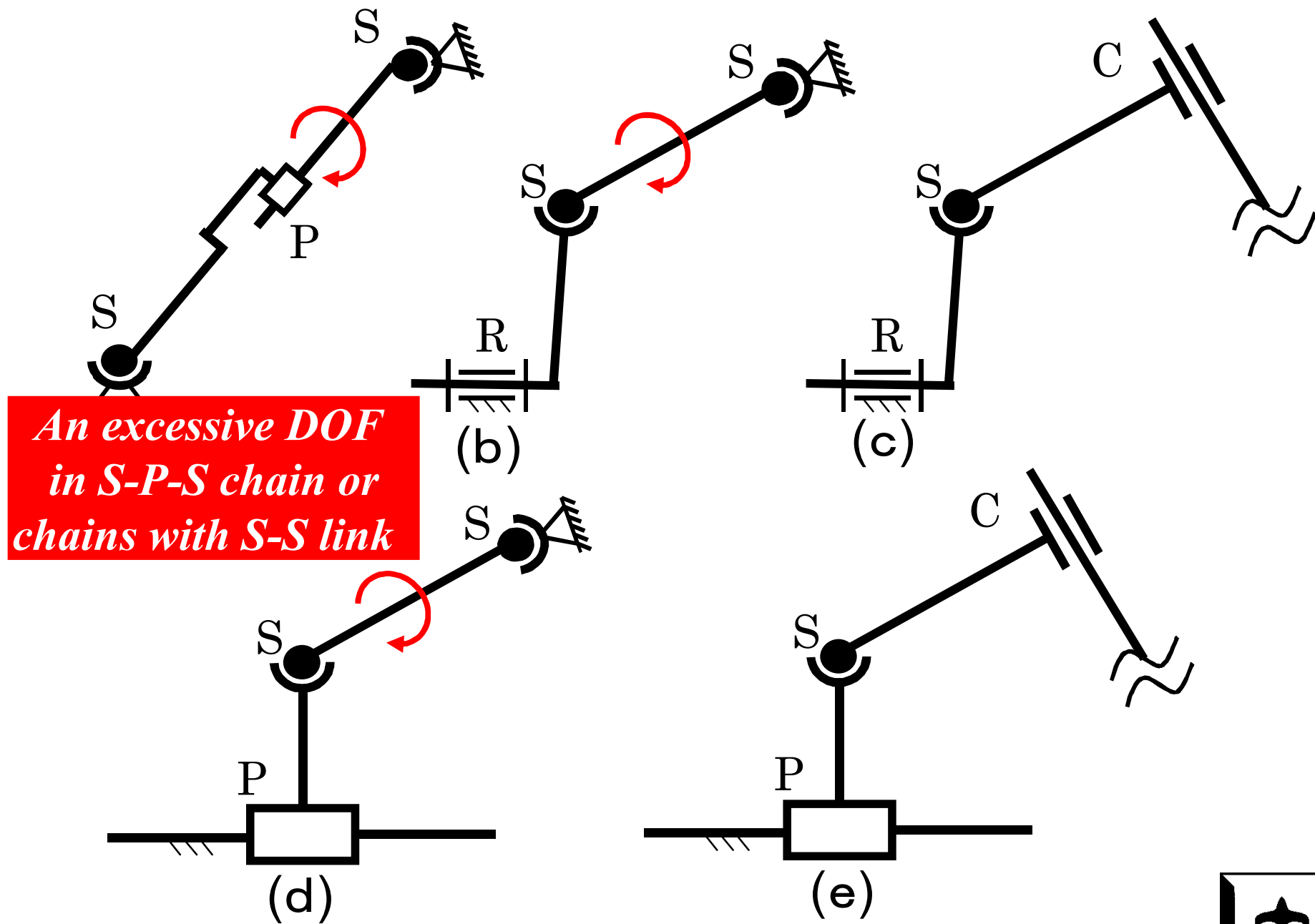


Next let's consider motion of spatial two adjacent links

Principle → Two adjacent links in which position and posture of all of links can be determined when position and posture of pairs at both ends

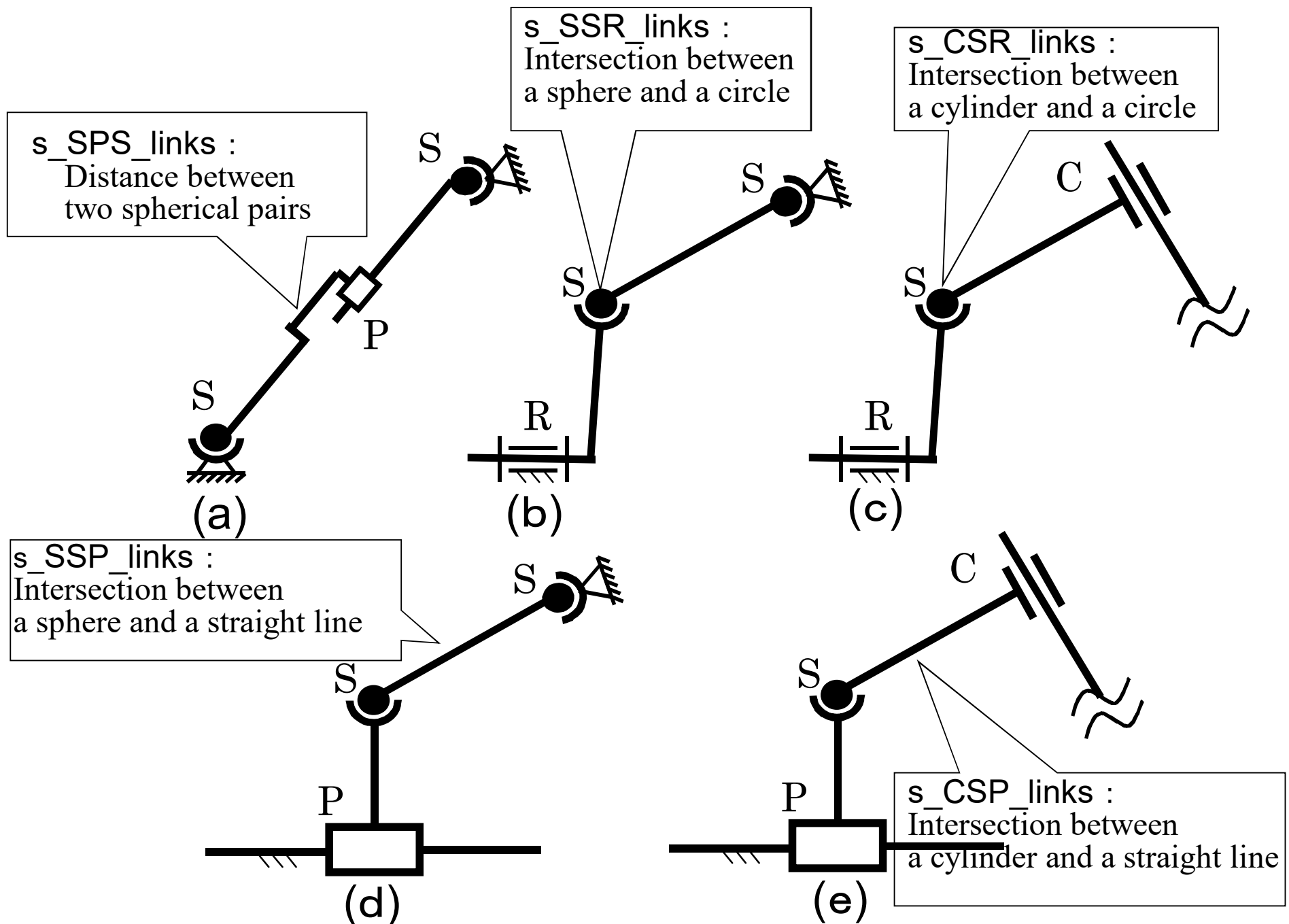
→ It means the link chain which becomes a structure when pairs at both ends become as fixed pairs.





Two adjacent links in spatial mechanisms

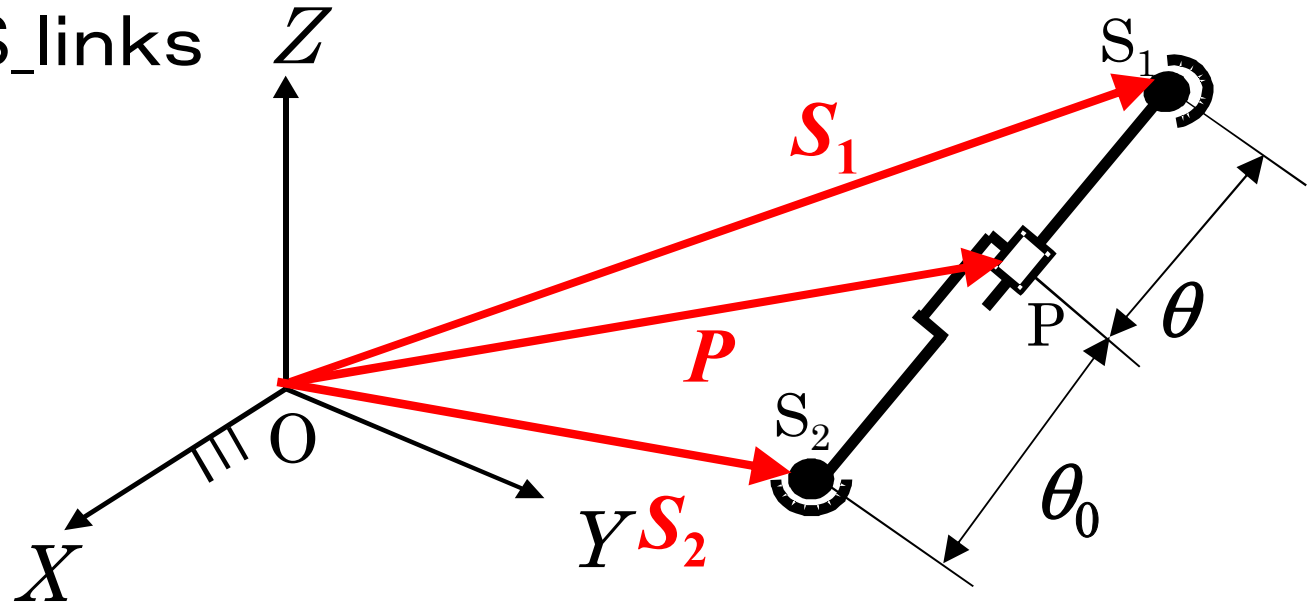




Kinematic analysis program for two adjacent links

Several examples:

(4) SPS links s_SPS_links



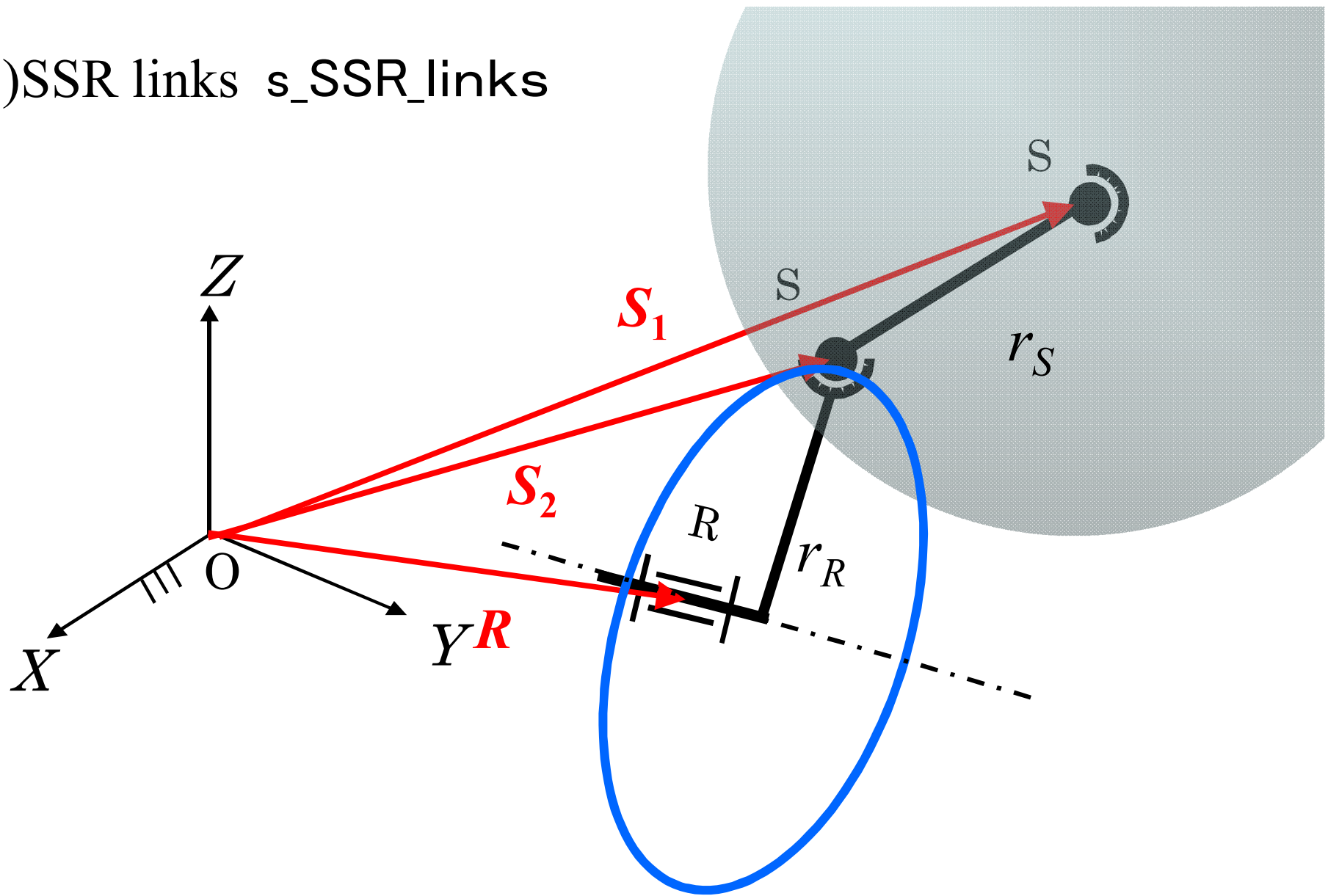
$$\theta + \theta_0 = |\mathbf{S}_2 - \mathbf{S}_1| = \sqrt{(\mathbf{S}_2 - \mathbf{S}_1) \bullet (\mathbf{S}_2 - \mathbf{S}_1)}$$

$$\theta = \sqrt{(\mathbf{S}_2 - \mathbf{S}_1) \bullet (\mathbf{S}_2 - \mathbf{S}_1)} - \theta_0$$

$$\dot{\theta} = \frac{(\dot{\mathbf{S}}_2 - \dot{\mathbf{S}}_1) \bullet (\mathbf{S}_2 - \mathbf{S}_1)}{\sqrt{(\mathbf{S}_2 - \mathbf{S}_1) \bullet (\mathbf{S}_2 - \mathbf{S}_1)}} = \frac{(\dot{\mathbf{S}}_2 - \dot{\mathbf{S}}_1) \bullet (\mathbf{S}_2 - \mathbf{S}_1)}{\theta + \theta_0}$$

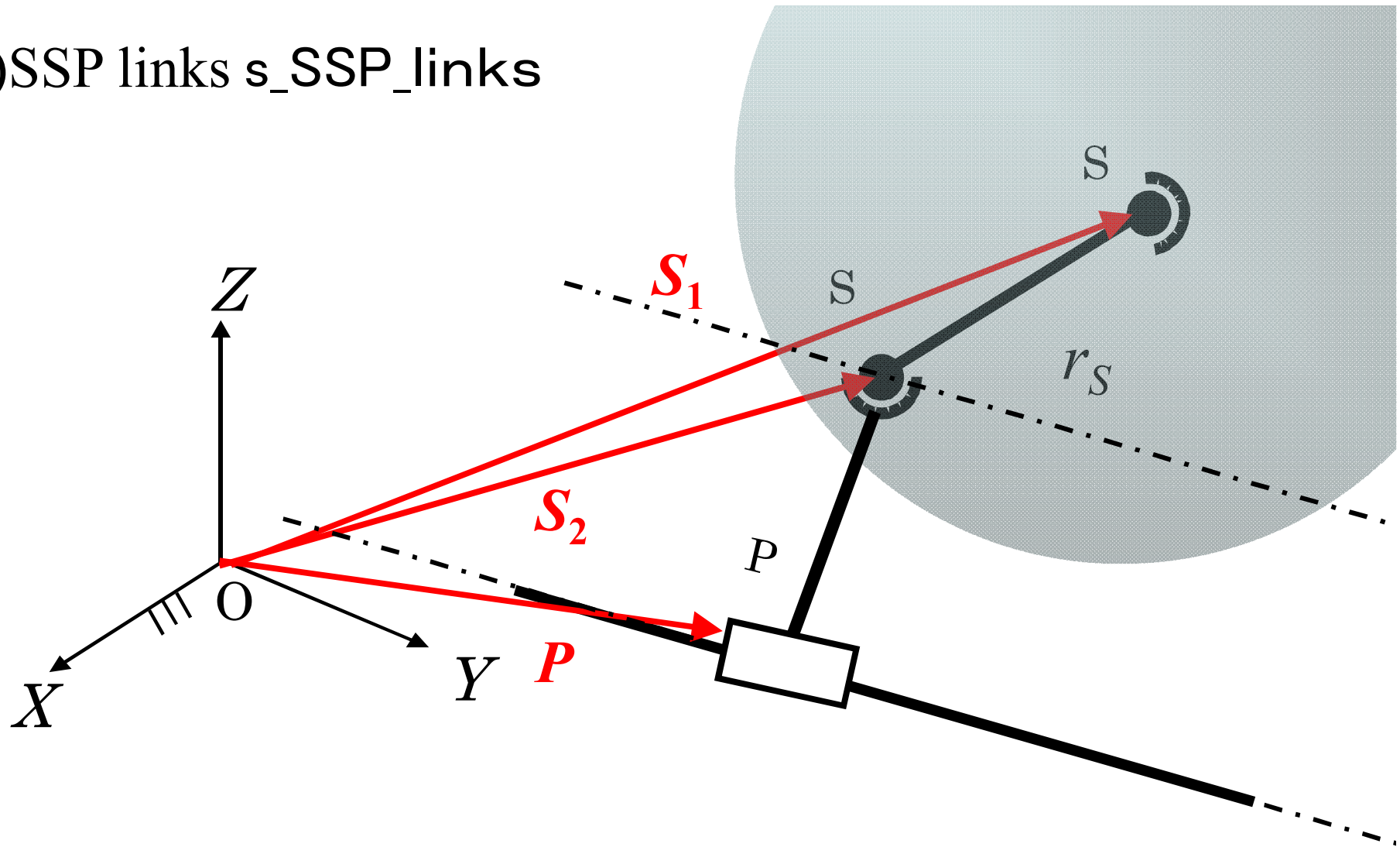
$$\ddot{\theta} = \frac{[(\ddot{\mathbf{S}}_2 - \ddot{\mathbf{S}}_1) \bullet (\mathbf{S}_2 - \mathbf{S}_1) + (\dot{\mathbf{S}}_2 - \dot{\mathbf{S}}_1) \bullet (\dot{\mathbf{S}}_2 - \dot{\mathbf{S}}_1)](\theta + \theta_0) - (\dot{\mathbf{S}}_2 - \dot{\mathbf{S}}_1) \bullet (\mathbf{S}_2 - \mathbf{S}_1)\dot{\theta}}{(\theta + \theta_0)^2}$$

(5)SSR links s_SSR_links



Position of pair S_2 can be calculated as an Intersection between a sphere with radius r_s and a circle with radius r_R

(6) SSP links s_SSP_links



Position of pair S_2 can be determined as an intersection between a sphere with radius r_s and a straight line parallel to the slider axis

```

//-----
//
//   ロール・ピッチ・ヨー角から座標変換行列を求める
//
//   RPY_ANGLES
//       : ロール・ピッチ・ヨー角とその微分
//   TT[3][3], dTT[3][3], ddTT[3][3]
//       : 座標変換行列とその微分
//-----

void RPYTT ( POSTURE RPY_ANGLES, double TT[3][3],
            double dTT[3][3], double ddTT[3][3] )
{
    double CR, SR, CP, SP, CY, SY;
    int i, j;
    double WA1[3][3], WA2[3][3], WA3[3][3],
           WB1[3][3], WB2[3][3], WB3[3][3],
           WC1[3][3], WC2[3][3], WC3[3][3];

    CR = cos( RPY_ANGLES.RPY.x );
    SR = sin( RPY_ANGLES.RPY.x );
    CP = cos( RPY_ANGLES.RPY.y );
    SP = sin( RPY_ANGLES.RPY.y );
    CY = cos( RPY_ANGLES.RPY.z );
    SY = sin( RPY_ANGLES.RPY.z );

    WA1[0][0] = 0.0;
    WA1[1][0] = -SR*SY + CR*SP*CY;
    WA1[2][0] = CR*SY + SR*SP*CY;
    WA1[0][1] = 0.0;
    WA1[1][1] = -SR*CY - CR*SP*SY;
    WA1[2][1] = CR*CY - SR*SP*SY;
    WA1[0][2] = 0.0;
    WA1[1][2] = -CR*CP;
    WA1[2][2] = -SR*CP;

    WA2[0][0] = -SP*CY;
    WA2[1][0] = SR*CP*CY;
    WA2[2][0] = -CR*CP*CY;
    WA2[0][1] = SP*SY;
    WA2[1][1] = -SR*CP*SY;
    WA2[2][1] = CR*CP*SY;
    WA2[0][2] = CP;
    WA2[1][2] = SR*SP;
    WA2[2][2] = -CR*SP;

```

```

    WA3[0][0] = -CP*SY;
    WA3[1][0] = CR*CY - SR*SP*SY;
    WA3[2][0] = SR*CY + CR*SP*SY;
    WA3[0][1] = -CP*CY;
    WA3[1][1] = -CR*SY - SR*SP*CY;
    WA3[2][1] = -SR*SY + CR*SP*CY;
    WA3[0][2] = 0.0;
    WA3[1][2] = 0.0;
    WA3[2][2] = 0.0;

```

```

    WB1[0][0] = 0.0;
    WB1[1][0] = -CR*SY - SR*SP*CY;
    WB1[2][0] = -SR*SY + CR*SP*CY;
    WB1[0][1] = 0.0;
    WB1[1][1] = -CR*CY + SR*SP*SY;
    WB1[2][1] = -SR*CY - CR*SP*SY;
    WB1[0][2] = 0.0;
    WB1[1][2] = SR*CP;
    WB1[2][2] = -CR*CP;

```

```

    WB2[0][0] = -CP*CY;
    WB2[1][0] = -SR*SP*CY;
    WB2[2][0] = CR*SP*CY;
    WB2[0][1] = CP*SY;
    WB2[1][1] = SR*SP*SY;
    WB2[2][1] = -CR*SP*SY;
    WB2[0][2] = -SP;
    WB2[1][2] = SR*CP;
    WB2[2][2] = -CR*CP;

```

```

    WB3[0][0] = -CP*CY;
    WB3[1][0] = -CR*SY - SR*SP*CY;
    WB3[2][0] = -SR*SY + CR*SP*CY;
    WB3[0][1] = CP*SY;
    WB3[1][1] = -CR*CY + SR*SP*SY;
    WB3[2][1] = -SR*CY - CR*SP*SY;
    WB3[0][2] = 0.0;
    WB3[1][2] = 0.0;
    WB3[2][2] = 0.0;

```

```

    WC1[0][0] = 0.0;
    WC1[1][0] = CR*CP*CY;
    WC1[2][0] = SR*CP*CY;
    WC1[0][1] = 0.0;
    WC1[1][1] = -CR*CP*SY;
    WC1[2][1] = -SR*CP*SY;

```

C language program of systematic kinematics analysis

```

WC2[0][0] = SP*SY;
WC2[1][0] = -SR*CP*SY;
WC2[2][0] = CR*CP*SY;
WC2[0][1] = SP*CY;
WC2[1][1] = -SR*CP*CY;
WC2[2][1] = CR*CP*CY;
WC2[0][2] = 0.0;
WC2[1][2] = 0.0;
WC2[2][2] = 0.0;

WC3[0][0] = 0.0;
WC3[1][0] = -SR*CY - CR*SP*SY;
WC3[2][0] = CR*CY - SR*SP*SY;
WC3[0][1] = 0.0;
WC3[1][1] = SR*SY - CR*SP*CY;
WC3[2][1] = -CR*SY - SR*SP*CY;
WC3[0][2] = 0.0;
WC3[1][2] = 0.0;
WC3[2][2] = 0.0;

TT[0][0] = CP*CY;
TT[1][0] = CR*SY + SR*SP*CY;
TT[2][0] = SR*SY - CR*SP*CY;
TT[0][1] = -CP*SY;
TT[1][1] = CR*CY - SR*SP*SY;
TT[2][1] = SR*CY + CR*SP*SY;
TT[0][2] = SP;
TT[1][2] = -SR*CP;
TT[2][2] = CR*CP;
for(i = 0; i < 3; i++){
    or(j = 0; j < 3; j++){
        dTT[i][j] = WA1[i][j]*RPY_ANGLES.DRPY.x
                    + WA2[i][j]*RPY_ANGLES.DRPY.y
                    + WA3[i][j]*RPY_ANGLES.DRPY.z;
        dTT[i][j] = WA1[i][j]*RPY_ANGLES.DDRPY.x
                    + WA2[i][j]*RPY_ANGLES.DDRPY.y
                    + WA3[i][j]*RPY_ANGLES.DDRPY.z
                    + WB1[i][j]*RPY_ANGLES.DRPY.x*
                      RPY_ANGLES.DRPY.x
                    + WB2[i][j]*RPY_ANGLES.DRPY.y*
                      RPY_ANGLES.DRPY.y
                    + WB3[i][j]*RPY_ANGLES.DRPY.z*
                      RPY_ANGLES.DRPY.z
                    + 2.0*WC1[i][j]*RPY_ANGLES.DRPY.x*
                      RPY_ANGLES.DRPY.y
                    + 2.0*WC2[i][j]*RPY_ANGLES.DRPY.y*
                      RPY_ANGLES.DRPY.z

```

```

                    + 2.0*WC3[i][j]*RPY_ANGLES.DRPY.z*
                      RPY_ANGLES.DRPY.x;
                }
            }
        }

//-----
//
// 座標変換行列に基づく3次元座標変換
//
// PM      : 動座標系上の位置ベクトル
// P0      : 動座標系原点の静止系上の位置ベクトル
//          とその微分
// TT[3][3], dTT[3][3], ddTT[3][3]
//          : 座標変換行列とその微分
// PF      : 静止系上の位置ベクトルとその微分
//
//-----
void TRANS( VECTOR3 PM, POSITION P0,
            double TT[3][3], double dTT[3][3], double ddTT[3][3],
            POSITION *PF )
{
    MPV3( TT, PM, &PF->P );
    MPV3( dTT, PM, &PF->DP );
    MPV3( ddTT, PM, &PF->DDP );

    PF->P.x = PF->P.x + P0.P.x;
    PF->P.y = PF->P.y + P0.P.y;
    PF->P.z = PF->P.z + P0.P.z;

    PF->DP.x = PF->DP.x + P0.DP.x;
    PF->DP.y = PF->DP.y + P0.DP.y;
    PF->DP.z = PF->DP.z + P0.DP.z;

    PF->DDP.x = PF->DDP.x + P0.DDP.x;
    PF->DDP.y = PF->DDP.y + P0.DDP.y;
    PF->DDP.z = PF->DDP.z + P0.DDP.z;
}

```

```
//-----
//
// 空間 球一直進一球対偶2連節の運動
//
// s1, s2 : 両端の球対偶の位置ベクトルとその微分
// p0 : s1からpまでの固定距離
// p : 直進対偶の位置ベクトルとその微分
// the : 直進対偶変位とその微分
//-----

int s_SPS_links ( POSITION s1, POSITION s2, double p0, POSITION *p, VARIABLE *the )
{
    double X, Y, Z, dX, dY, dZ, ddX, ddY, ddZ, TH, dTH, ddTH, w1, w2, w3;

    X = s2.P.x - s1.P.x;
    Y = s2.P.y - s1.P.y;
    Z = s2.P.z - s1.P.z;
    dX = s2.DP.x - s1.DP.x;
    dY = s2.DP.y - s1.DP.y;
    dZ = s2.DP.z - s1.DP.z;
    ddX = s2.DDP.x - s1.DDP.x;
    ddY = s2.DDP.y - s1.DDP.y;
    ddZ = s2.DDP.z - s1.DDP.z;

    TH = sqrt( X*X + Y*Y + Z*Z );
    if( fabs( TH ) < 1.0e-30 ) {
        printf( "++ Error in s_SPS_links (特異状態です)¥n");
        return ERROR;
    }
    dTH = ( dX*X + dY*Y + dZ*Z )/TH;
    ddTH = ( ( ddX*X + dX*dX + ddY*Y + dY*dY + ddZ*Z + dZ*dZ )*TH -
        ( dX*X + dY*Y + dZ*Z )*dTH )/( TH*TH );
    w1 = p0/TH;
    w2 = -p0*dTH/( TH*TH );
    w3 = -p0*( ddTH - 2.0*dTH*dTH*TH )/( TH*TH*TH*TH );
    p->P.x = w1*X + s1.P.x;
    p->P.y = w1*Y + s1.P.y;
    p->P.z = w1*Z + s1.P.z;

    p->DP.x = w2*X + w1*dX + s1.DP.x;
    p->DP.y = w2*Y + w1*dY + s1.DP.y;
    p->DP.z = w2*Z + w1*dZ + s1.DP.z;

    p->DDP.x = w3*X + 2.0*w2*dX + w1*ddX + s1.DDP.x;
    p->DDP.y = w3*Y + 2.0*w2*dY + w1*ddY + s1.DDP.y;
    p->DDP.z = w3*Z + 2.0*w2*dZ + w1*ddZ + s1.DDP.z;
    the->v = TH - p0;
    the->dv = dTH;
    the->ddv = ddTH;

    return SUCCESS;
}
```

```
//-----
//
// 空間 球一球一直進対偶2連節の運動
//
// s1 : 端の球対偶の位置ベクトルとその微分
// rs : 2球対偶間の距離
// g : 直進対偶の直動ガイドを表す空間直線上の1点の
// 位置ベクトルとその微分
// TLT[3][3], dTLT[3][3], ddTLT[3][3]
// : g3を原点, 直進対偶の直動ガイドを表す空間直線を
// Z軸とする動座標系の座標変換行列とその微分(転置)
// s2p : 中央の球対偶の位置ベクトル
// (直進対偶に固定した動座標系)
// minv : 機構の反転の指示インジケータ
// the : 直進対偶の変位とその微分
// p : 直進対偶の位置ベクトルとその微分
// s2 : 中央の球対偶の位置ベクトルとその微分
//-----

int s_SSP_links ( POSITION s1, double rs, POSITION g,
    double TLT[3][3], double dTLT[3][3], double ddTLT[3][3],
    VECTOR3 s2p, int minv, VARIABLE *the,
    POSITION *p, POSITION *s2 )
{
    VECTOR3 UVLF, PLS1F, dUVLF, dPLS1F, ddUVLF, ddPLS1F,
    ww, dww, ddww;
    double a, da, dda, b, db, ddb, d, rootd, w1, w2, w3, w4, w5;

    UVLF.x = TLT[0][2];
    UVLF.y = TLT[1][2];
    UVLF.z = TLT[2][2];
    PLS1F.x = g.P.x - s1.P.x;
    PLS1F.y = g.P.y - s1.P.y;
    PLS1F.z = g.P.z - s1.P.z;

    VIV3 ( UVLF, PLS1F, &a );
    a = a + s2p.z;

    MPV3 ( TLT, s2p, &ww );
    VIV3 ( ww, PLS1F, &b );
    b = b*2.0 + PLS1F.x*PLS1F.x + PLS1F.y*PLS1F.y
    + PLS1F.z*PLS1F.z + s2p.x*s2p.x + s2p.y*s2p.y + s2p.z*s2p.z - rs*rs;

    d = a*a - b;
```

```

if( d<0 ){
    printf("++ Error in s_SSP_links (連鎖が構成できません.) %n");
    return ERROR;
}
rootd = sqrt( d );
dUVLF.x = dTLT[0][2];
dUVLF.y = dTLT[1][2];
dUVLF.z = dTLT[2][2];
dPLS1F.x = g.DP.x - s1.DP.x;
dPLS1F.y = g.DP.y - s1.DP.y;
dPLS1F.z = g.DP.z - s1.DP.z;
ddUVLF.x = ddTLT[0][2];
ddUVLF.y = ddTLT[1][2];
ddUVLF.z = ddTLT[2][2];
ddPLS1F.x = g.DDP.x - s1.DDP.x;
ddPLS1F.y = g.DDP.y - s1.DDP.y;
ddPLS1F.z = g.DDP.z - s1.DDP.z;

VIV3 ( dUVLF, PLS1F, &w1 );
VIV3 ( UVLF, dPLS1F, &da );
da = da + w1;
VIV3 ( ddUVLF, PLS1F, &w1 );
VIV3 ( dUVLF, dPLS1F, &w2 );
VIV3 ( UVLF, ddPLS1F, &dda );
dda = dda + w1 + 2.0*w2;

MPV3 ( dTLT, s2p, &dww );
MPV3 ( ddTLT, s2p, &ddww );
VIV3 ( dPLS1F, PLS1F, &w1 );
VIV3 ( dww, PLS1F, &w2 );
VIV3 ( ww, dPLS1F, &w3 );

db = 2.0*( w1 + w2 + w3 );
VIV3 ( ddPLS1F, PLS1F, &w1 );
VIV3 ( dPLS1F, dPLS1F, &w2 );
VIV3 ( ddww, PLS1F, &w3 );
VIV3 ( dww, dPLS1F, &w4 );
VIV3 ( ww, ddPLS1F, &w5 );
ddb = 2.0*( w1 + w2 + w3 + 2.0*w4 + w5 );

```

```

p->P.x = the->v*UVLF.x + g.P.x;
p->P.y = the->v*UVLF.y + g.P.y;
p->P.z = the->v*UVLF.z + g.P.z;
p->DP.x = the->dv*UVLF.x + the->v*dUVLF.x + g.DP.x;
p->DP.y = the->dv*UVLF.y + the->v*dUVLF.y + g.DP.y;
p->DP.z = the->dv*UVLF.z + the->v*dUVLF.z + g.DP.z;
p->DDP.x = the->ddv*UVLF.x + 2.0*the->dv*dUVLF.x
    + the->v*ddUVLF.x + g.DDP.x;
p->DDP.y = the->ddv*UVLF.y + 2.0*the->dv*dUVLF.y
    + the->v*ddUVLF.y + g.DDP.y;
p->DDP.z = the->ddv*UVLF.z + 2.0*the->dv*dUVLF.z
    + the->v*ddUVLF.z + g.DDP.z;

s2->P.x = p->P.x + ww.x;
s2->P.y = p->P.y + ww.y;
s2->P.z = p->P.z + ww.z;
s2->DP.x = p->DP.x + dww.x;
s2->DP.y = p->DP.y + dww.y;
s2->DP.z = p->DP.z + dww.z;
s2->DDP.x = p->DDP.x + ddww.x;
s2->DDP.y = p->DDP.y + ddww.y;
s2->DDP.z = p->DDP.z + ddww.z;

return SUCCESS;
}

```

You can download these programs from

WEB-site:

<http://www.dynamics.mep.titech.ac.jp/japanese/download.html>

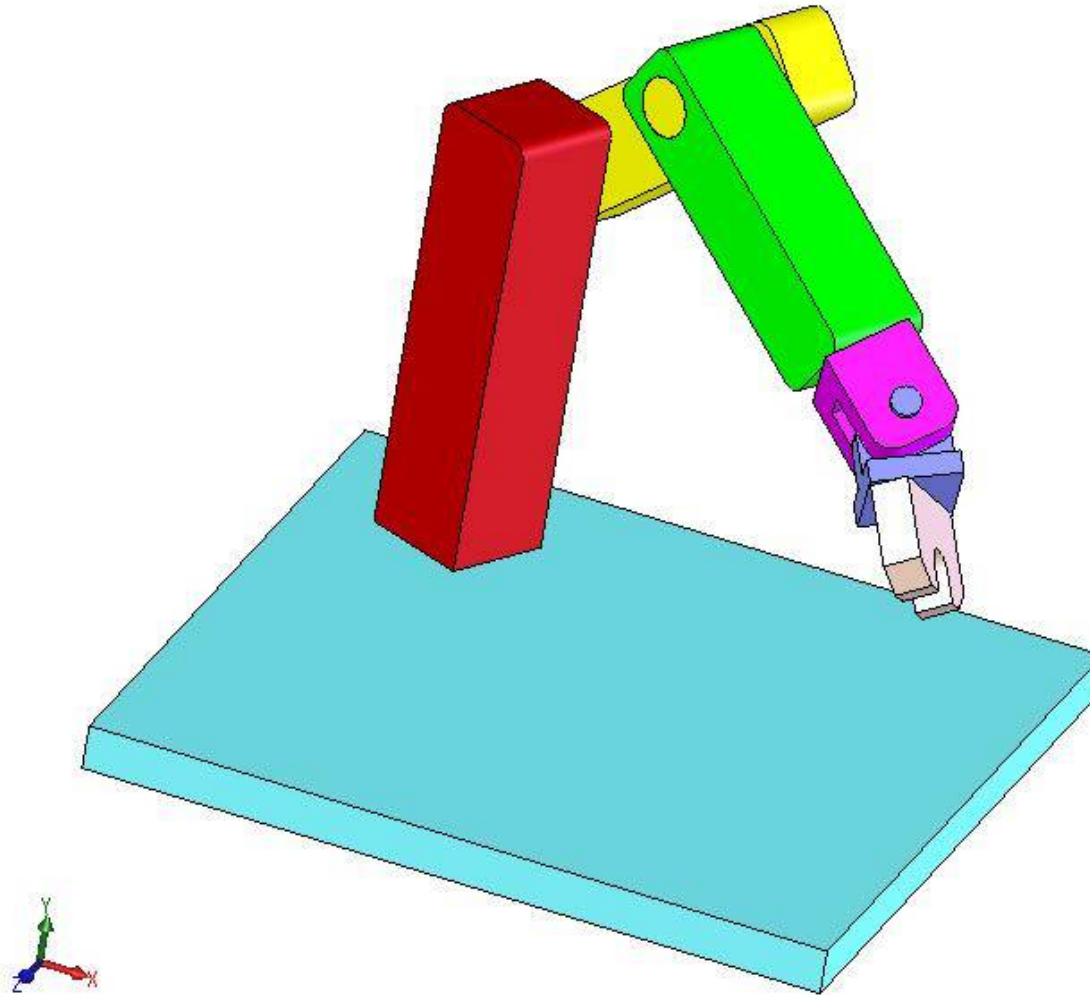
```

the->ddv = -dda - ( 2.0*(2.0*dda*a + 2.0*da*da
    - ddb )*d - ( 2.0*da*a - db )*( 2.0*da*a - db ) )
    /( 4.0*d*rootd );

```

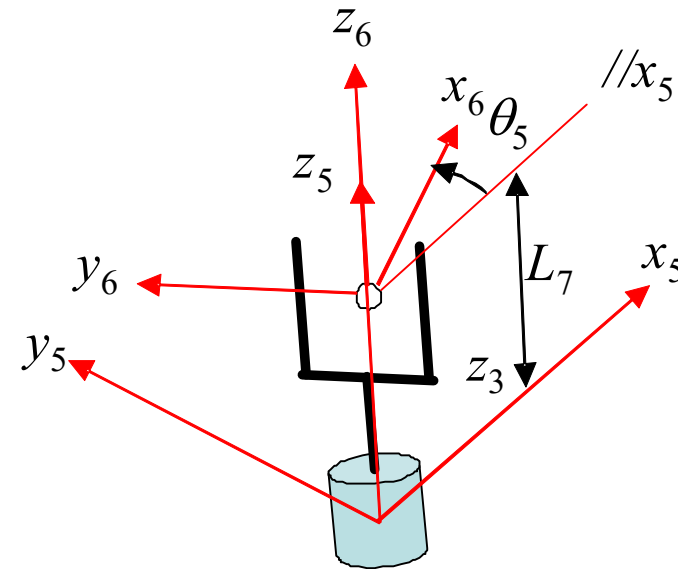
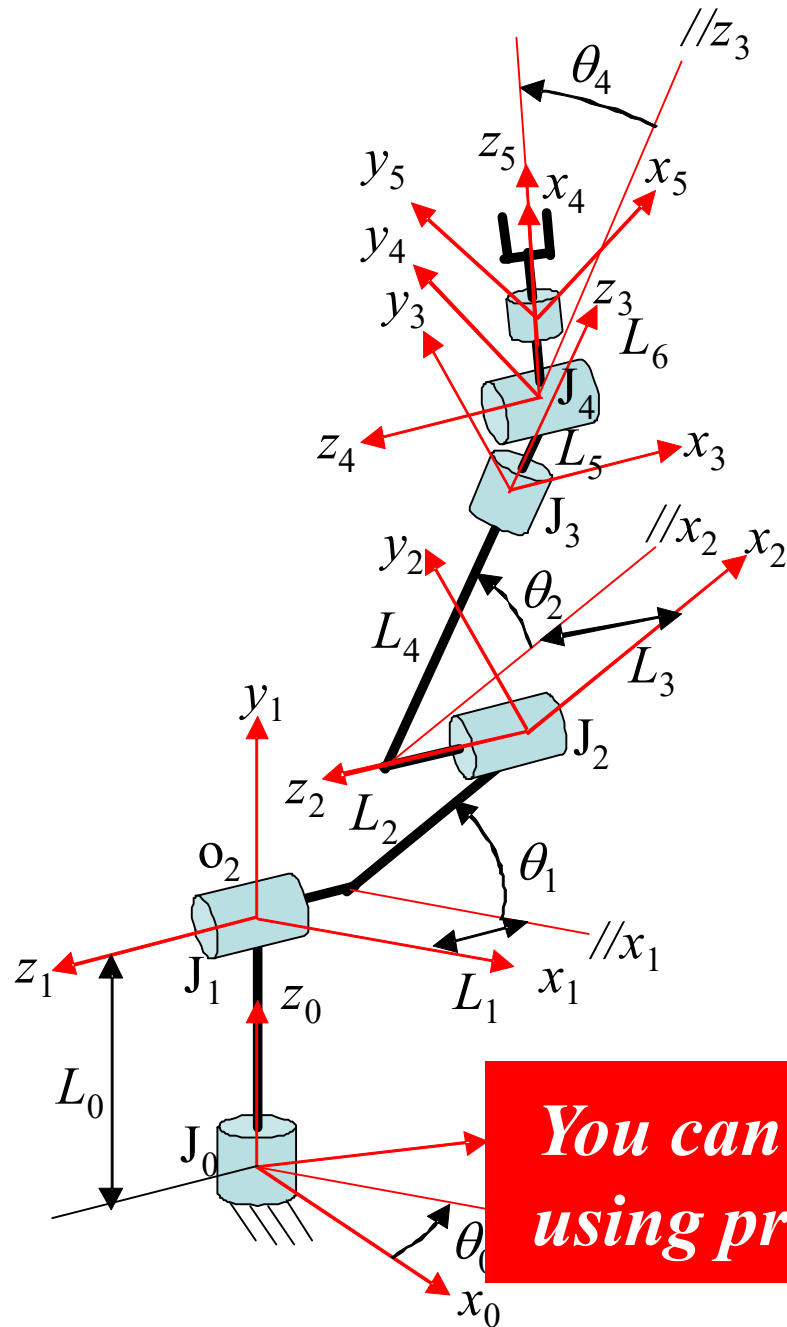

Examples of analysis:

(1) PUMA type spatial serial manipulator



Spatial serial mechanism with only revolute pairs and 6 DOF

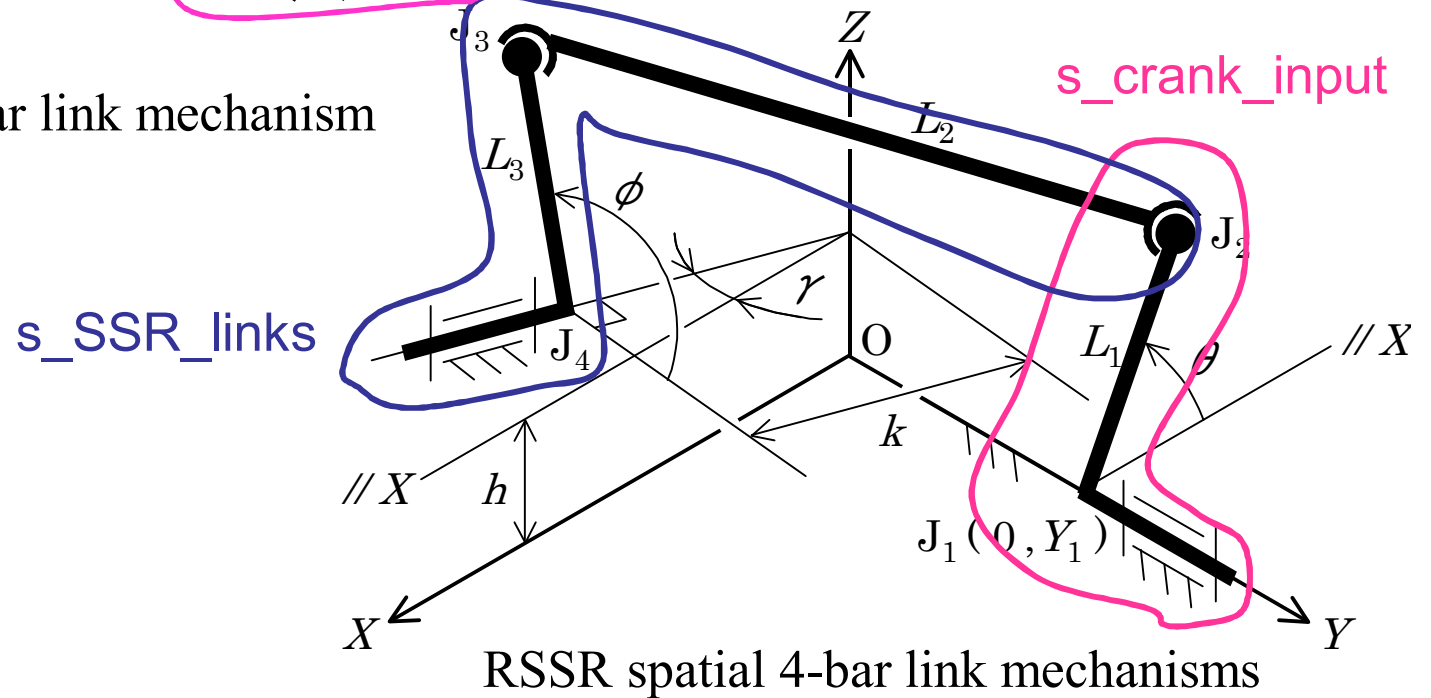
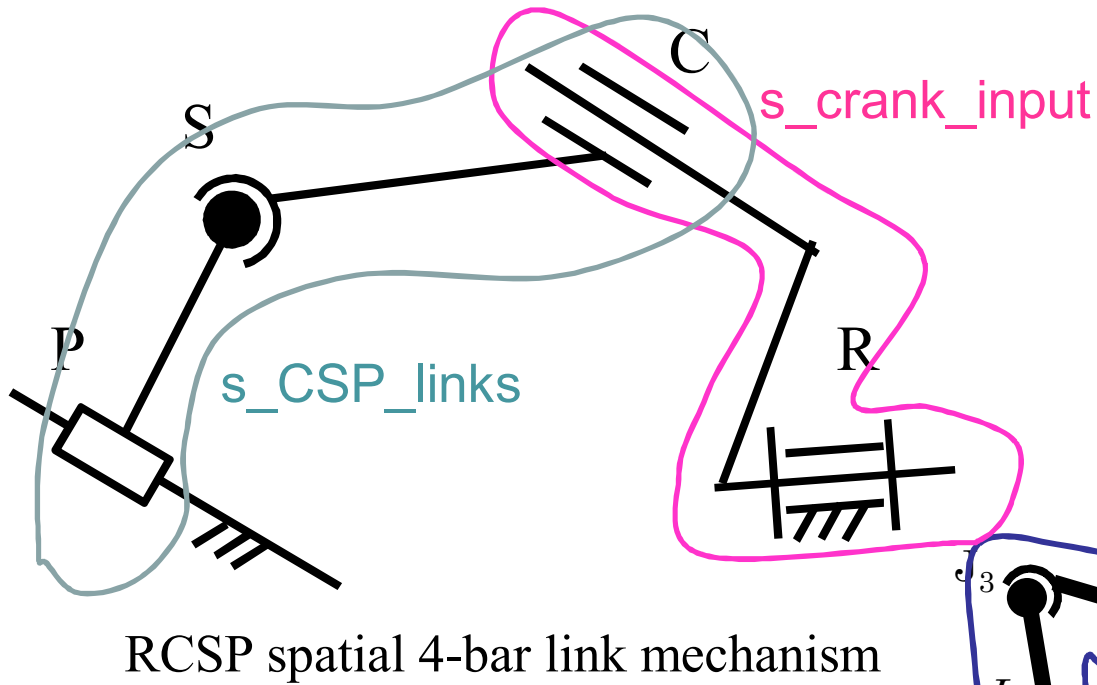




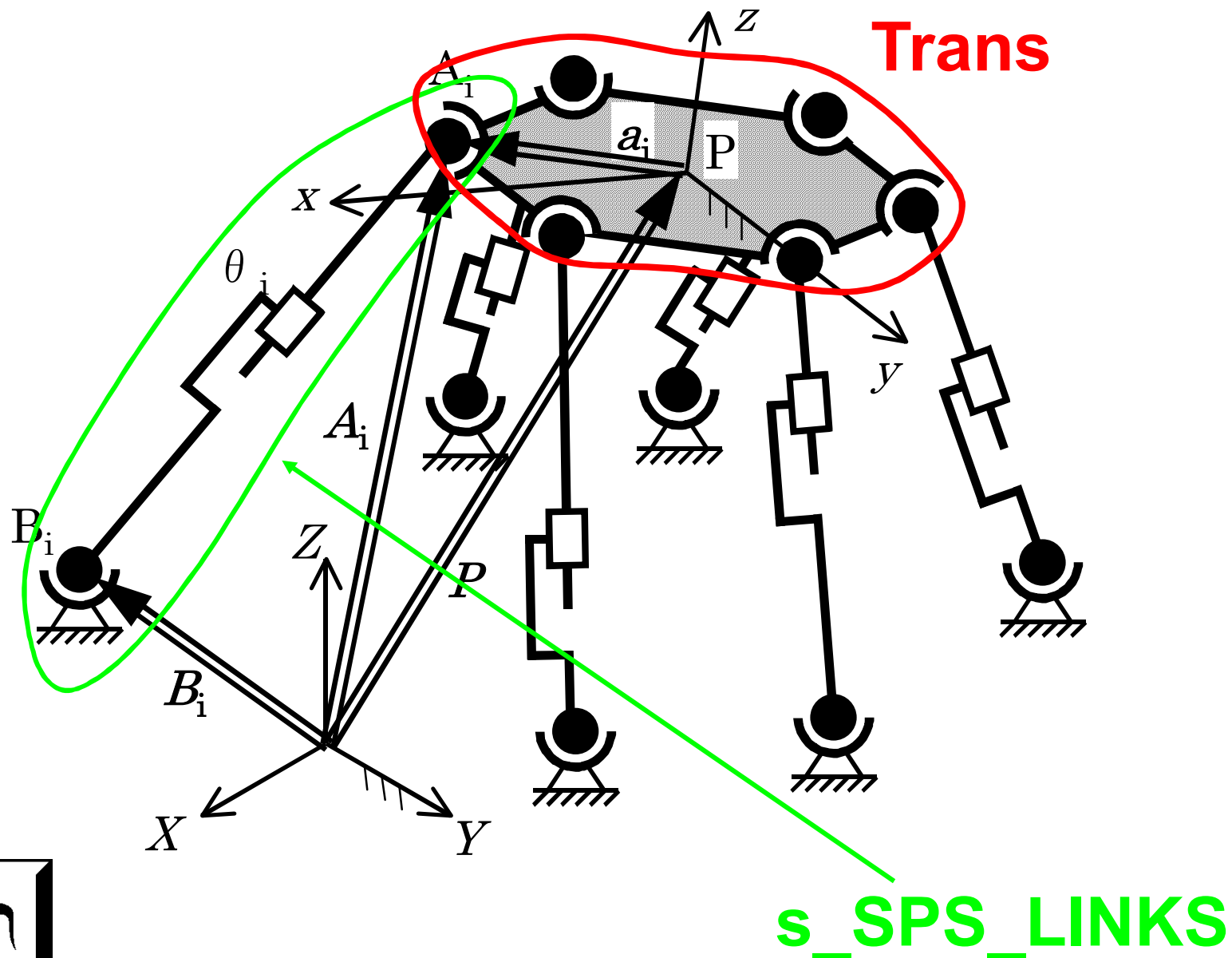
Location of coordinate systems

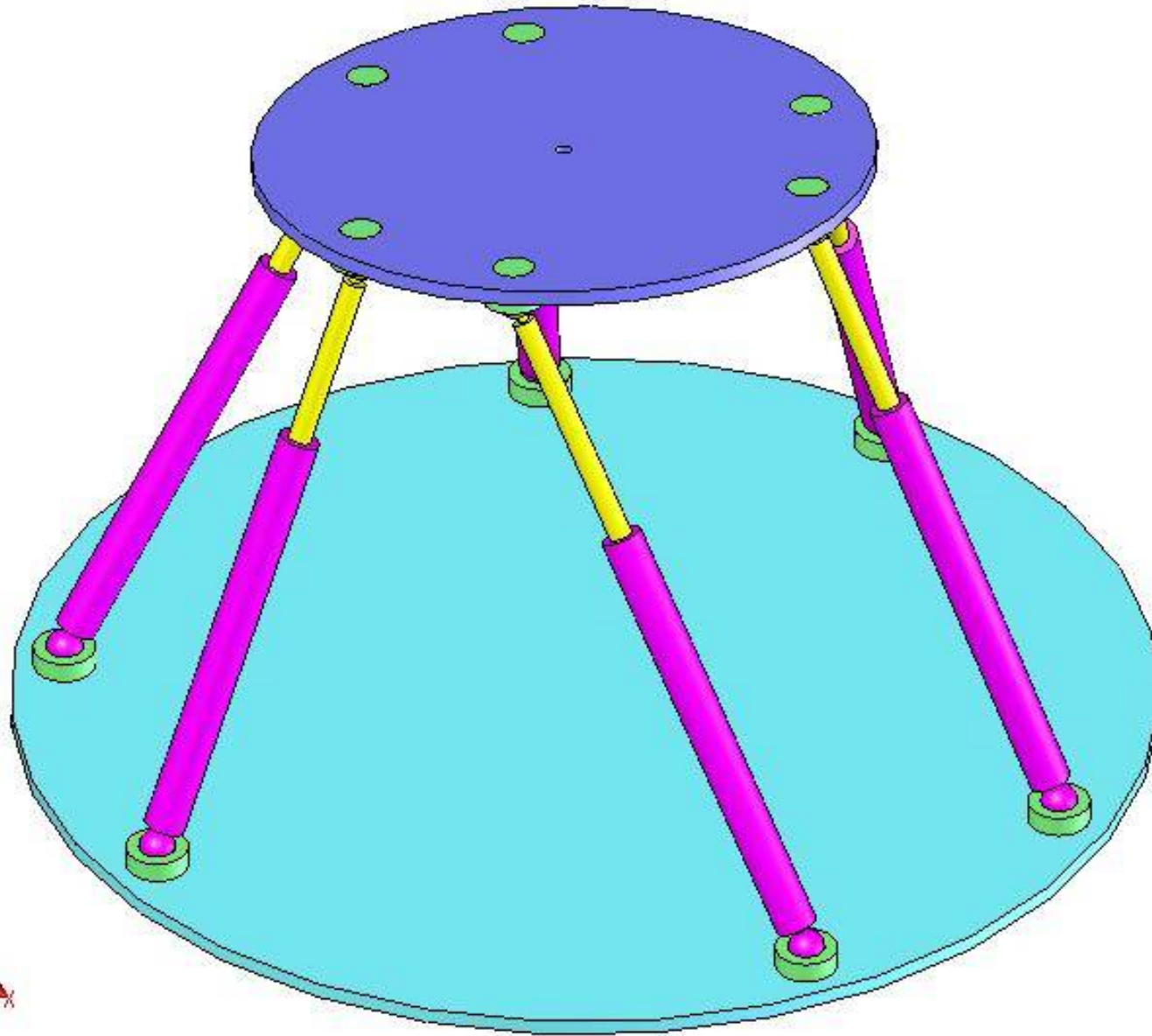
You can calculate forward kinematics by using program :s_crank_input only.

(2) Spatial 4-bar link mechanisms with 1 DOF



(3) Inverse kinematics of Stewart platform manipulator





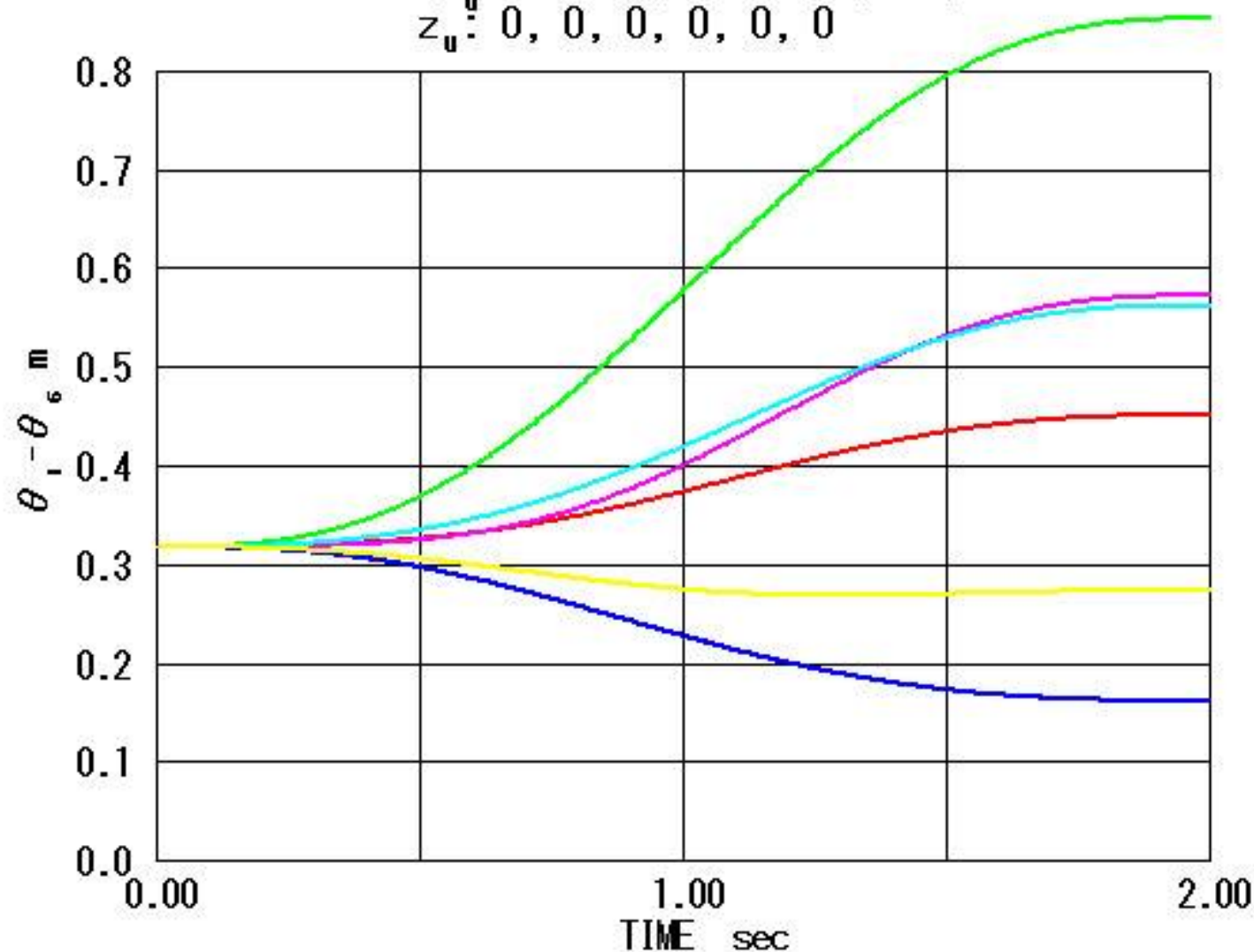
Example

スチュワートプラットフォームマニピュレータの逆運動学解析 with RPIKSTEWARTP.BAS

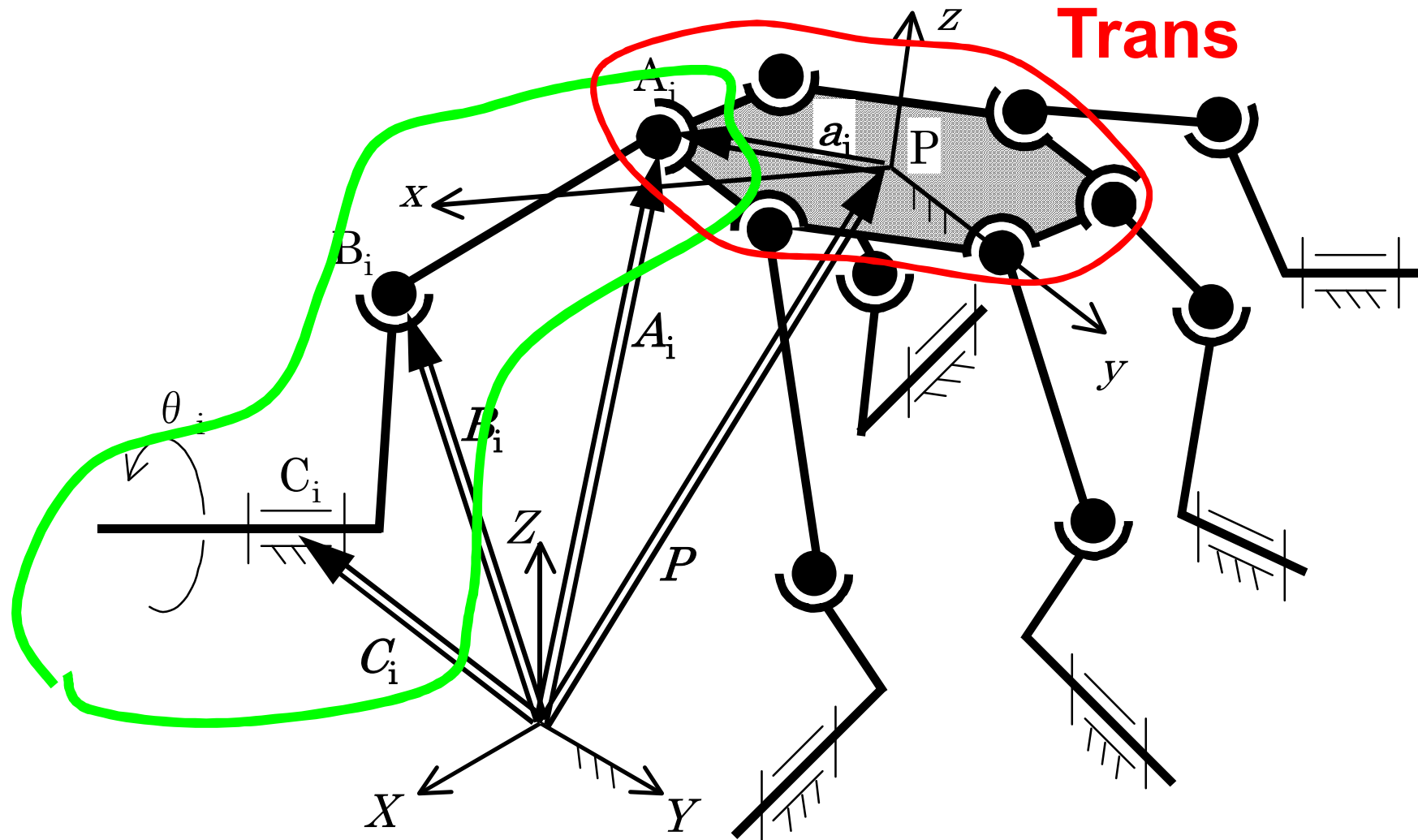
入力関節変位

Example of calculation

$R : 1, 1, 1, 1, 1, 1$
 $\psi : 20, 100, 140, 220, 260, -20$
 $Z : 0, 0, 0, 0, 0, 0$
 $R^L : .5, .5, .5, .5, .5, .5$
 $\psi^L : 40, 80, 160, 200, -80, -40$
 $z_u : 0, 0, 0, 0, 0, 0$



(4) Inverse kinematics of 6RSS spatial parallel manipulator



s_SSR_LINKS

By changing two adjacent links

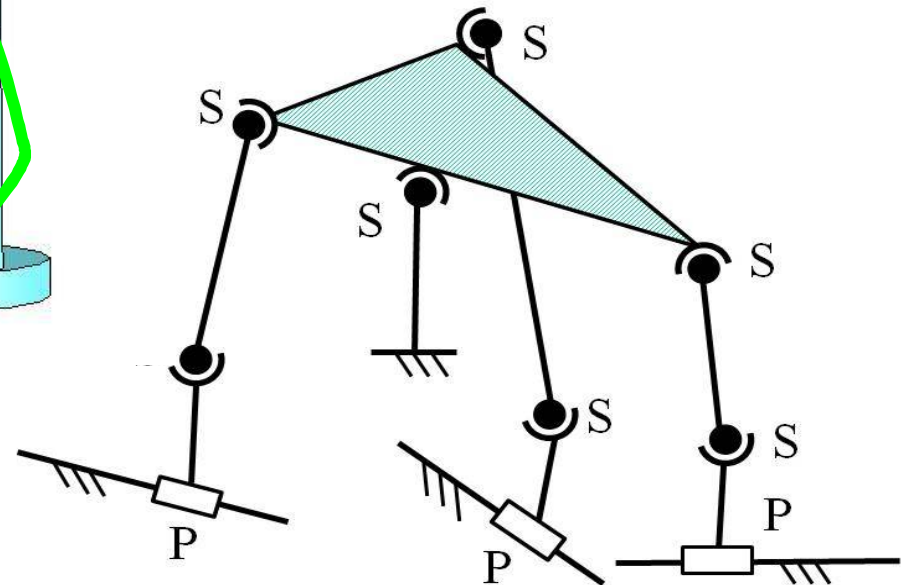
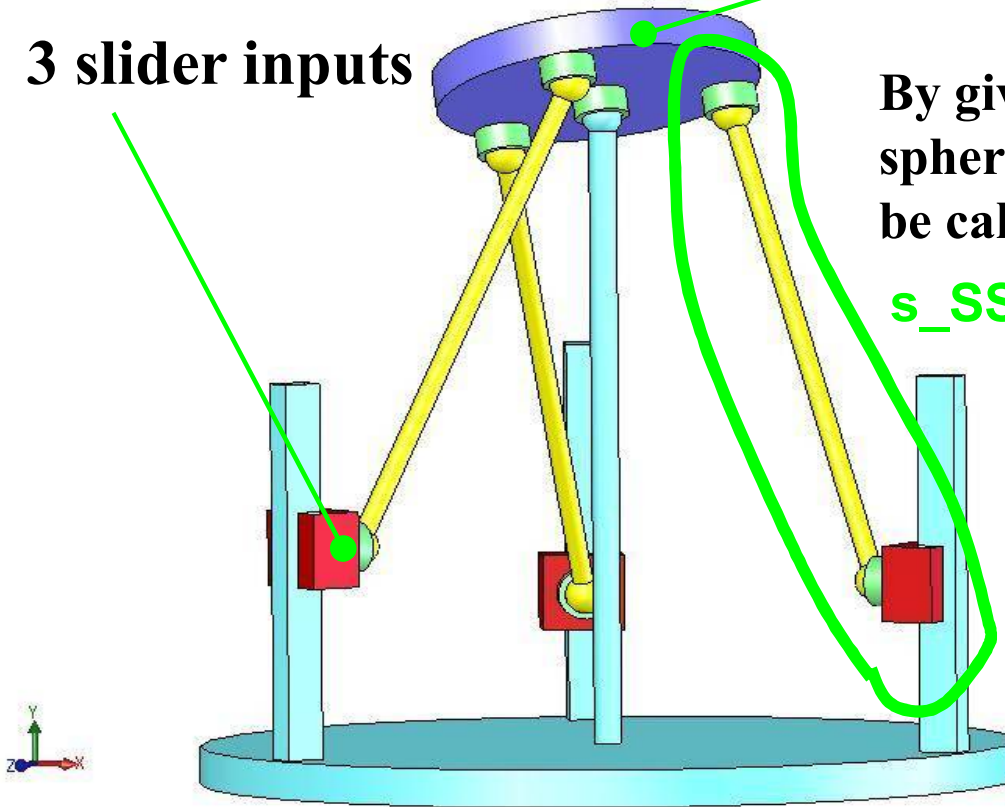
(5) Inverse kinematics of a camera head driving mechanism with 3 DOF

A camera is mounted on a moving platform and its 3 posture angles can be controlled.

3 slider inputs

By giving 3 posture angles, positions of spherical pairs on the moving platform can be calculated.

s_SSP_links



You can analyze various spatial link mechanisms by using the offered programs.

Principal analyses:

- (1) Spatial angular motion expressed by coordinate transformation matrix*
- (2) Crank/slider input links*
- (3) Two adjacent links*
- (4) Coupler point motion as coordinate transformation*





4. Concluding remarks

As an available tool to analyze or design link mechanisms is expanded to spatial mechanisms.

(1) Spatial angular motion can be expressed with coordinate transformation.

(2) Principal analysis:

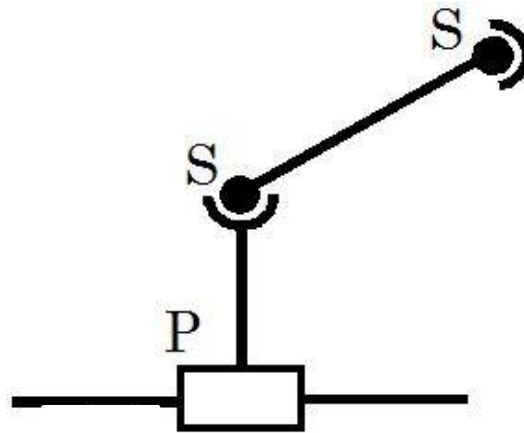
Input link/Two adjacent links/Coupler point

(3) Not only displacement but also velocity and acceleration can be calculated for various mechanisms

(4) Students are expected to experience to analyze some mechanisms with the offered programs.

Homework 2

Derive the position vector of spherical pair S_2 in SSP links when position and posture of prismatic pair P and position of pair S_1 at both ends and mechanism constants are given.



The result will be summarized in A4 size PDF with less than 5 pages and sent to Prof. Iwatsuki via T2SCHOLA by May 2, 2023.